

# Natural Refrigerant Training Summit

Building a Sustainable Workforce

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## Introduction to CO2 Systems Design

Ignacio Varela Chaparro

Kysor Warren Epta US



# Natural Refrigerant Training Summit

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# Who We Are

A 501c3 nonprofit working to create a sustainable future for supermarket refrigeration by removing barriers to natural refrigerant adoption.




**160+**  
member  
companies



**55K+**  
food retail  
locations



# Goals

-  Build a sustainable technician workforce
-  Increase funding for natural refrigerant equipment
-  Improve technology options, education, and awareness

## What are Natural Refrigerants?

**CO<sub>2</sub>**

**R744**  
Carbon Dioxide

**C<sub>3</sub>H<sub>8</sub>**

**R290**  
Propane

**NH<sub>3</sub>**

**R717**  
Ammonia



NORTH AMERICAN  
**Sustainable Refrigeration Council**



Council

# INTRODUCTION TO CO2 SYSTEMS DESIGN

Ignacio Varela Chaparro  
R&D & New Technologies Manager



**KYSOR WARREN**  
eptarefrigeration

- ***CO2 AS REFRIGERANT***
- ***SAFETY***
- ***DESIGN OVERVIEW***
- ***SUBSYSTEMS & COMPONENTS***
- ***CONTROLLERS***



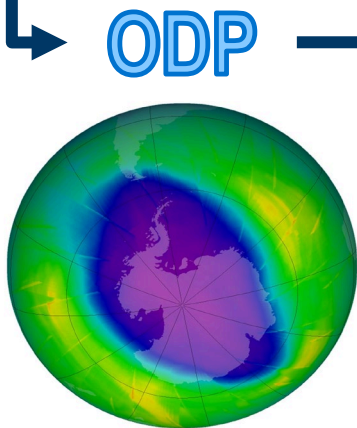
# REFRIGERANTS TRANSITION

Early Refrigeration

CO<sub>2</sub>  
NH<sub>3</sub>  
Naturals

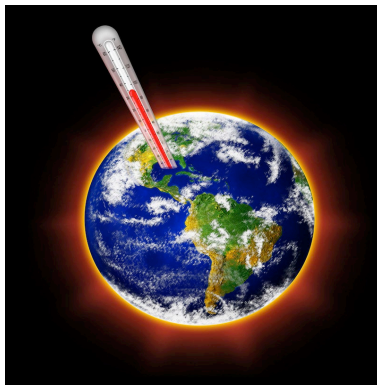
Today's Refrigeration

CO<sub>2</sub>  
NH<sub>3</sub>  
R290  
Naturals



**CFC & HCFC**  
Ozone Layer Damage  
R-22 / R-502

GWP



**HFC**  
Global Warming  
R- 404A / R-507A  
R448A / R449A

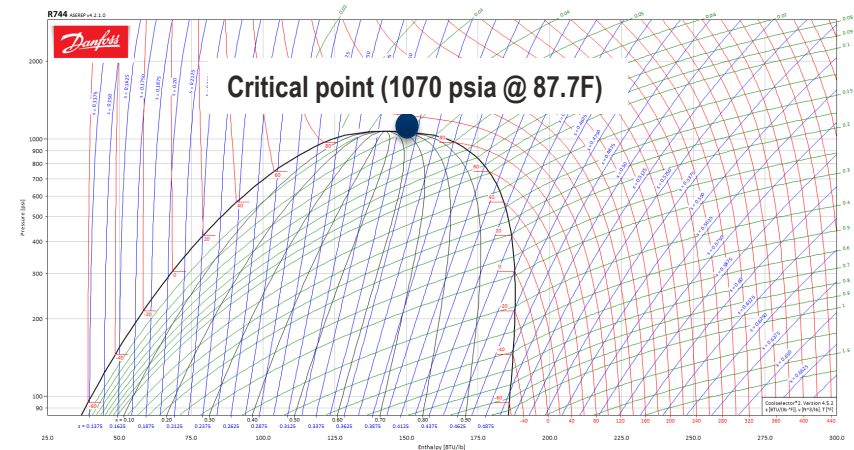
PFAS

Per- and Polyfluoroalkyl Substances



**HFO**  
Water and Air Contamination  
R471A / R454C / R455A

- Natural component of air
- Concentration in atmosphere approximately 400 ppm (0.04%)
- Colorless
- Odorless
- Heavier than air
- High Pressure Refrigerant (above 1000 psia @ 87F)
- High volumetric capacity
  - Smaller components & less refrigerant
- Low cost refrigerant
- Not subject to phase-out
- ASHRAE Number – R744
- Hi triple point – **Initial charge with VAPOR (above 150 psi)**
  - **Do not charge CO2 liquid if pressure is below 100 psi**

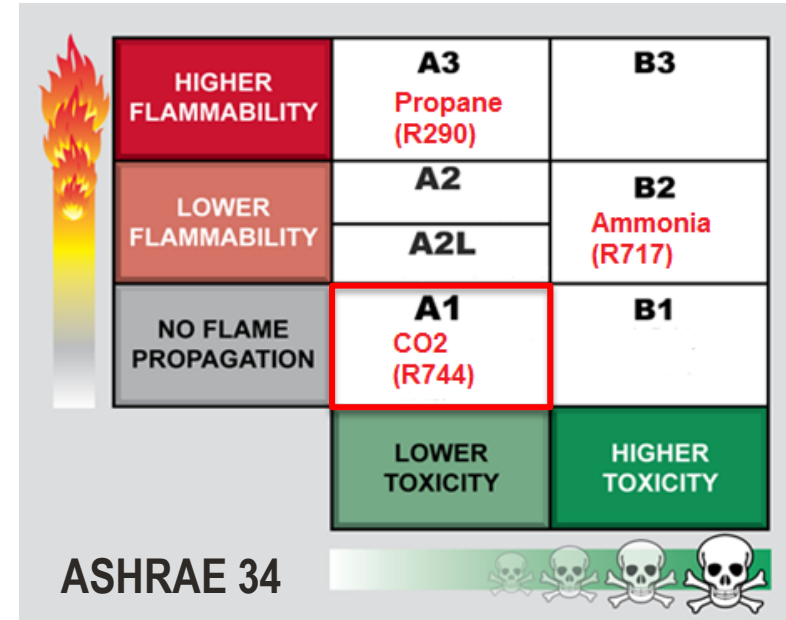


# REFRIGERANTS COMPARISON

## CO2 AND OTHER COMMON REFRIGERANTS

COMMERCIAL NAME	REFRIGERANT NUMBER	SAFETY GROUP	GWP (100 YEARS)	ODP
Solstice N40	R448A	A1	1273 <sup>(2)</sup>	0
Opteon® XP40.	R449A	A1	1397 <sup>(1)</sup>	0
	R404A	A1	3922 <sup>(1)</sup>	0
	R407A	A1	2107 <sup>(1)</sup>	0
	R22	A1	1810 <sup>(1)</sup>	0.055 <sup>(1)</sup>
Carbon Dioxide	R744	A1	1	0
Ammonia	R717	B2	0	0
Propane	R290	A3	3	0

(1) Linde / (2) Honeywell



New refrigerants

A1 - R471A,

A2L - R454C & R455A

CO2 (R744) is the only refrigerant that operates in transcritical mode





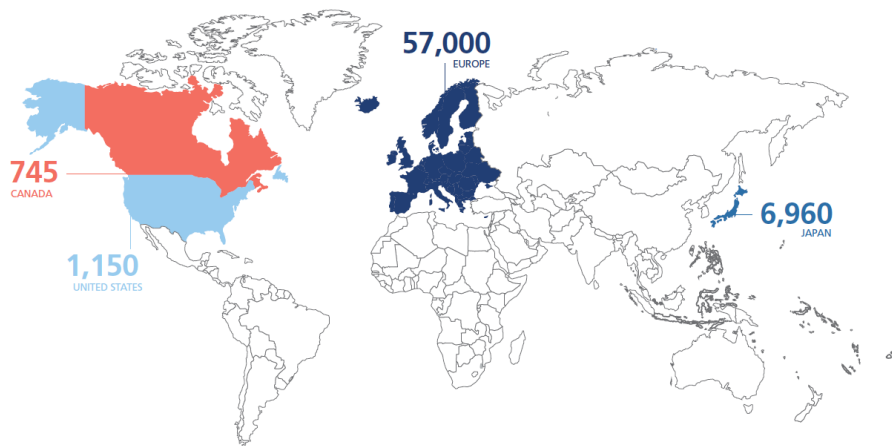
2022 Edition

# Natural Refrigerants: State of the Industry

CO<sub>2</sub> Installations are increasing at a fast pace.

## Transcritical CO<sub>2</sub> Installations in Major Regions

(stores and industrial sites, as of December 2022)



### Regulations enforcement

- California - CARB
- Washington – Department of Ecology
- **USA – EPA – Final Rule effective 01/2026-2027**
- Europe – F- Gas

## OCTOBER 5<sup>TH</sup> 2023 - FINAL RULE

Refrigeration, Air Conditioning, and Heat Pump Systems*			
Sector	Systems	Global Warming Potential Limit or Prohibited Substances	Installation Compliance Date <sup>5</sup>
Cold storage warehouses	With 200 or more lb refrigerant charge, excluding high temperature side of cascade system	150	January 1, 2026
	With less than 200 lb refrigerant charge	300	January 1, 2026
	High temperature side of cascade system	300	January 1, 2026
Retail food - supermarkets	With 200 or more lb refrigerant charge, excluding high temperature side of cascade system	150	January 1, 2027
	With less than 200 lb refrigerant charge	300	January 1, 2027
	High temperature side of cascade systems	300	January 1, 2027
Retail food - remote condensing units	With 200 or more lb refrigerant charge, excluding high temperature side of cascade system	150	January 1, 2026
	With less than 200 lb refrigerant charge	300	January 1, 2026
	High temperature side of cascade system	300	January 1, 2026

**SAFETY**



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# SAFETY

## BASIC SAFETY PRINCIPLES

- It is recommended for all staff working on this type of installation to use personal protective equipment, at minimum:

- ✓ **Gloves**
- ✓ **Glasses**
- ✓ **Safety shoes**

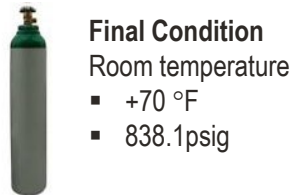
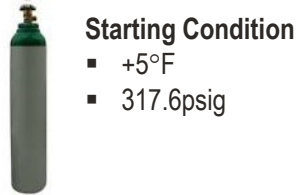


- Be aware of system pressure
  - **CO2 systems operate at much higher pressure than HFCs**
- Utilize tools and materials rated for proper pressures
  - **System operating pressures range 150 – 1500 psi**
- Avoid trapping CO2 in the system
  - **CO2 refrigerant expands quickly with temperature increase**
- Provide PRV for overpressure protection
- Avoid skin contact of refrigerant due to risk of frost or burn
- Prevent excessive vibration in piping
- Only authorized and **trained personnel** should perform maintenance or repairs on these systems

## AVOID TRAPPING CO<sub>2</sub>

- Liquid CO<sub>2</sub> has a very high coefficient of thermal expansion
- An increase in temperature will cause high increase in pressure

### Example:



### Effect of 1F temperature increase at different conditions

-20F (200psig) to -19F (204psig) = 4psig

+20F (407psig) to +21F (413psig) = 6psig

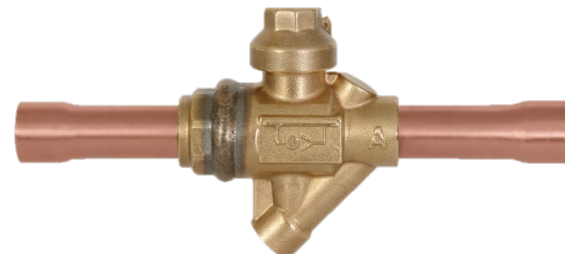
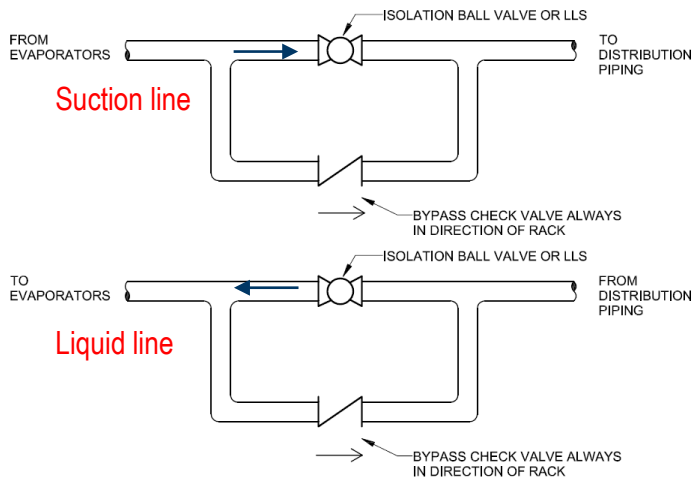
+50F (638psig) to +51F (647psig) = 9psig

+69F (827psig) to +70F (838psig) = 11psig

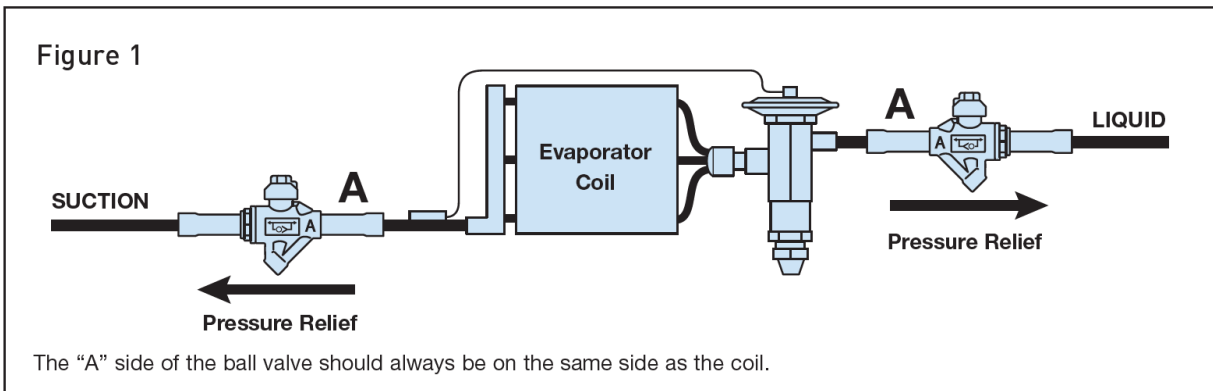
# SAFETY

## AVOID TRAPPING CO2 – PRVs & VALVE STATIONS

Check valves in both liquid and suction lines shall direct flow back to compressor rack

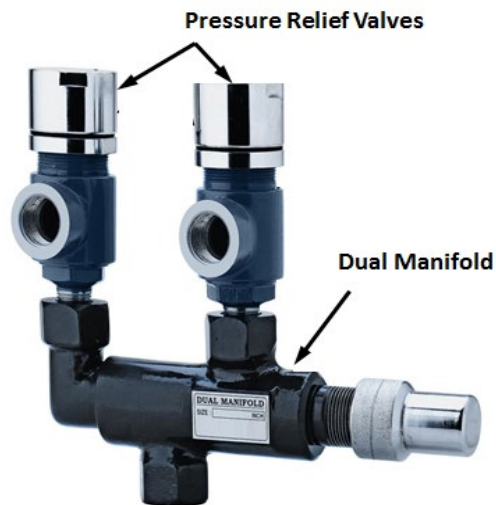


Ball valve with integrated check bypass



Relief valves are sealed by the manufacturer and should never be adjusted in the field.

Comply with local and national codes for installation of PRVs.



## ANSI/ASHRAE Standard 15-2022

### 9.4 Pressure Relief Protection

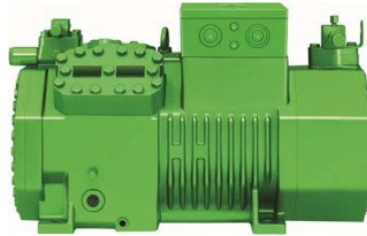
**9.4.1** *Refrigerating systems shall be protected by a pressure relief device or other approved means to safely relieve pressure due to fire or other abnormal conditions.*

**9.4.2** *Pressure vessels shall be protected in accordance with Section 9.7. Pressure relief devices are acceptable if they either bear a nameplate or are directly marked with a “UV” or “VR” symbol signifying compliance with ASME Boiler and Pressure Vessel Code 15, Section VIII.*

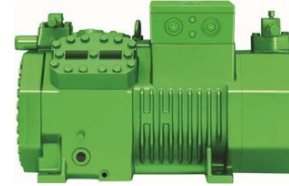
PRVs rated operation:

- Vapor
- Liquid / Vapor

### MT



### LT



CO2 compressors come with PRV installed but those protect the compressor only, not the system  
PRV setpoint on compressors are normally higher than system design pressure

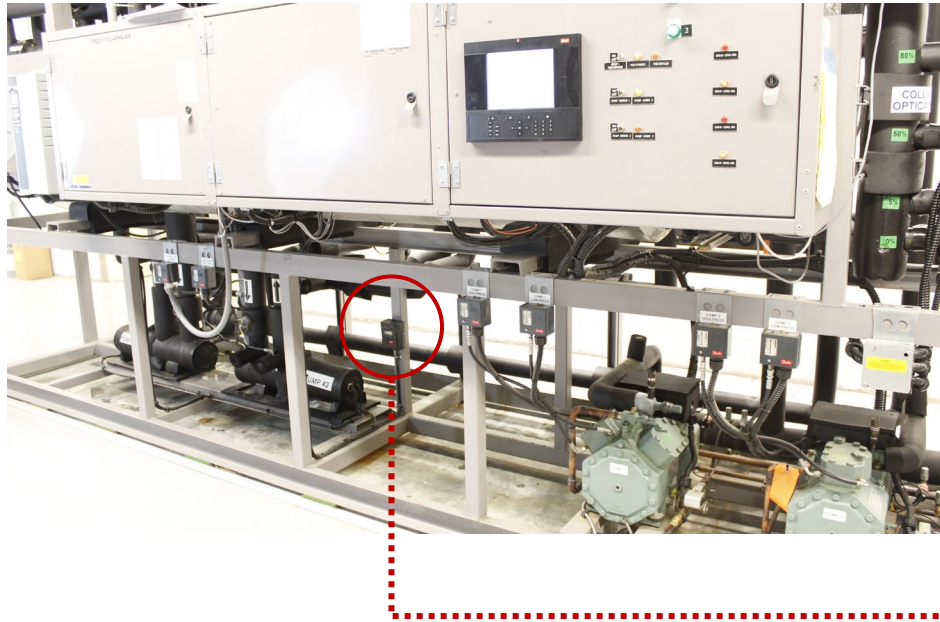


# SAFETY

## LEAK DETECTION SYSTEM

### LEAK DETECTION EQUIPMENT

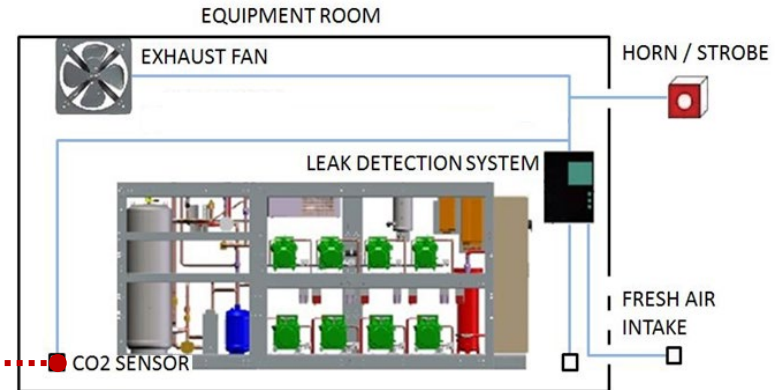
- Install leak detector sensors that trigger an alarm when the concentration level exceeds predefined CO2 concentration (ppm)
- CO2 is heavier than air, the sensors and ventilators should be located close to the floor (CO2=44 g/mol vs AIR 28 g/mol)
- Check ventilation system requirements, **code compliance**



CO2 concentration detection as low as 5 to 20 ppm

Leak detection systems

- ✓ Emerson RLDS/MRLDS
- ✓ Danfoss GDC IR 10000/40000
- ✓ Vaisala CO2 leak detector



### CO2 - GENERAL INSTRUCTIONS

- CO2 is unique among natural refrigerants for having good safety characteristics
  - ☑ **It is non-flammable**
  - ☑ **Non-explosive**
  - ☑ **Relatively non-toxic**
- If enough concentration of carbon dioxide in an enclosed space, over a certain period of time can cause asphyxiation
- According to ASHRAE 34, a **CO2 concentration of 1000 ppm** is the maximum limit to ensure the comfort for the occupants

OSHA lists 5000 ppm as the TLV-TWA (Threshold Limit Value – Time Weighted Average).

CO2 Concentration		Symptoms and effects on humans
%	ppm	
0.04%	400	Normal concentration in the atmosphere
0.1%	1000	Recommended upper limit for comfortable indoor air quality
0.5%	5000	8 hours – long term exposure limit TLV-TWA (OSHA)
1%	10,000	Drowsiness
1.5%	15,000	10 minutes – short term exposure limit TLV-TWA
2%	20,000	50% increase of breathing
3%	30,000	OSHA STEL 10 minute TWA exposure, 100% increase of breathing
5%	50,000	IDLH – Immediate Danger to Life or Health
10%	100,000	Lowest lethal concentration, few minutes exposure causes unconsciousness
30%	300,000	Brief exposure causes unconsciousness and death

# DESIGN OVERVIEW



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# CO2 REFRIGERATION

SYSTEM SOLUTION CONFIGURATIONS – HFC CHARGE REDUCTION

Concept	Medium Temp	Low Temp	HFC Charge*
Direct Expansion (Conventional)	DX HFC	DX HFC	100%
HFC / CO2 Cascade	DX HFC	DX CO2	70%
HFC / Glycol / CO2 Cascade	SC Glycol	DX CO2	15% – 30%
HFC / Glycol / CO2	SC Glycol	CO2 LOF	15% – 30%
HFC / CO2 Hybrid	LOF*** CO2	DX CO2	15% – 30%
Natural Ref** / CO2 Hybrid	LOF*** CO2	DX CO2	0%
CO2 Transcritical Booster	DX CO2	DX CO2	0%

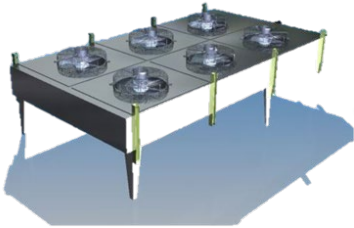
\*HFC Refrigerant use

\*\* NH3/CO2 Mostly used for industrial applications

\*\*\* LOF: Liquid Overfeed = CO2 pumped

# SYSTEM SOLUTIONS

HFC DIRECT EXPANSION MT DX, LT DX



**Condenser**



**HFC Rack**

100% HFC

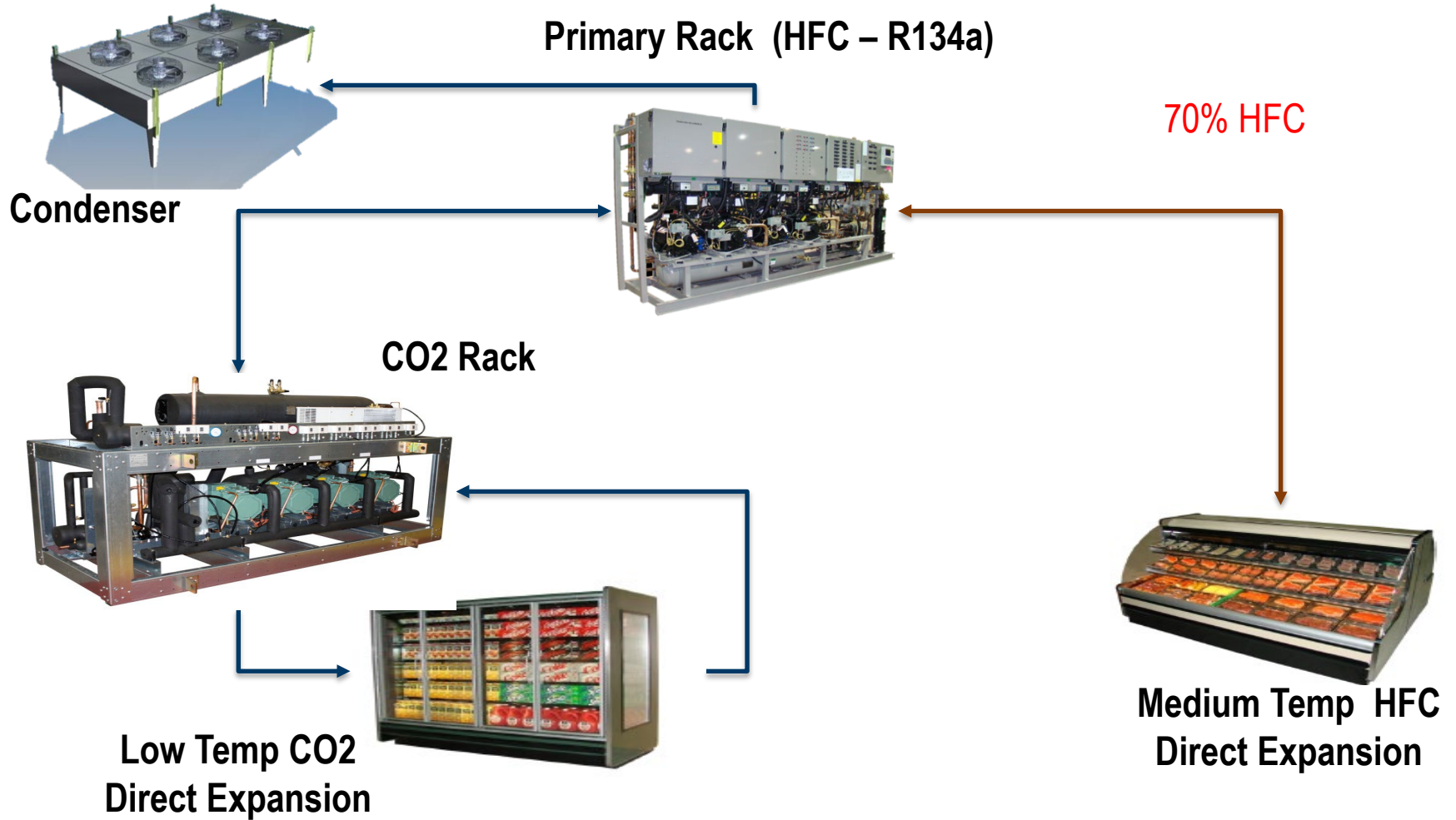


**Low Temp HFC  
Direct Expansion**

**Medium Temp HFC  
Direct Expansion**

# SYSTEM SOLUTIONS

CASCADE SYSTEM HFC MT / CO2 LT DX

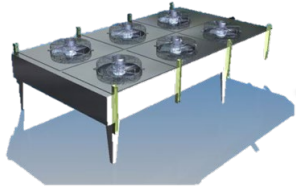


# SYSTEM SOLUTIONS

CASCADE SYSTEM HFC / GLYCOL MT / CO2 LT DX

## Primary Rack (HFC,NH3,HC)

0% - 30% HFC



Condenser

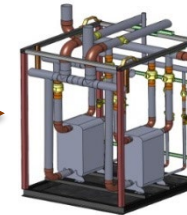
## CO2 Rack



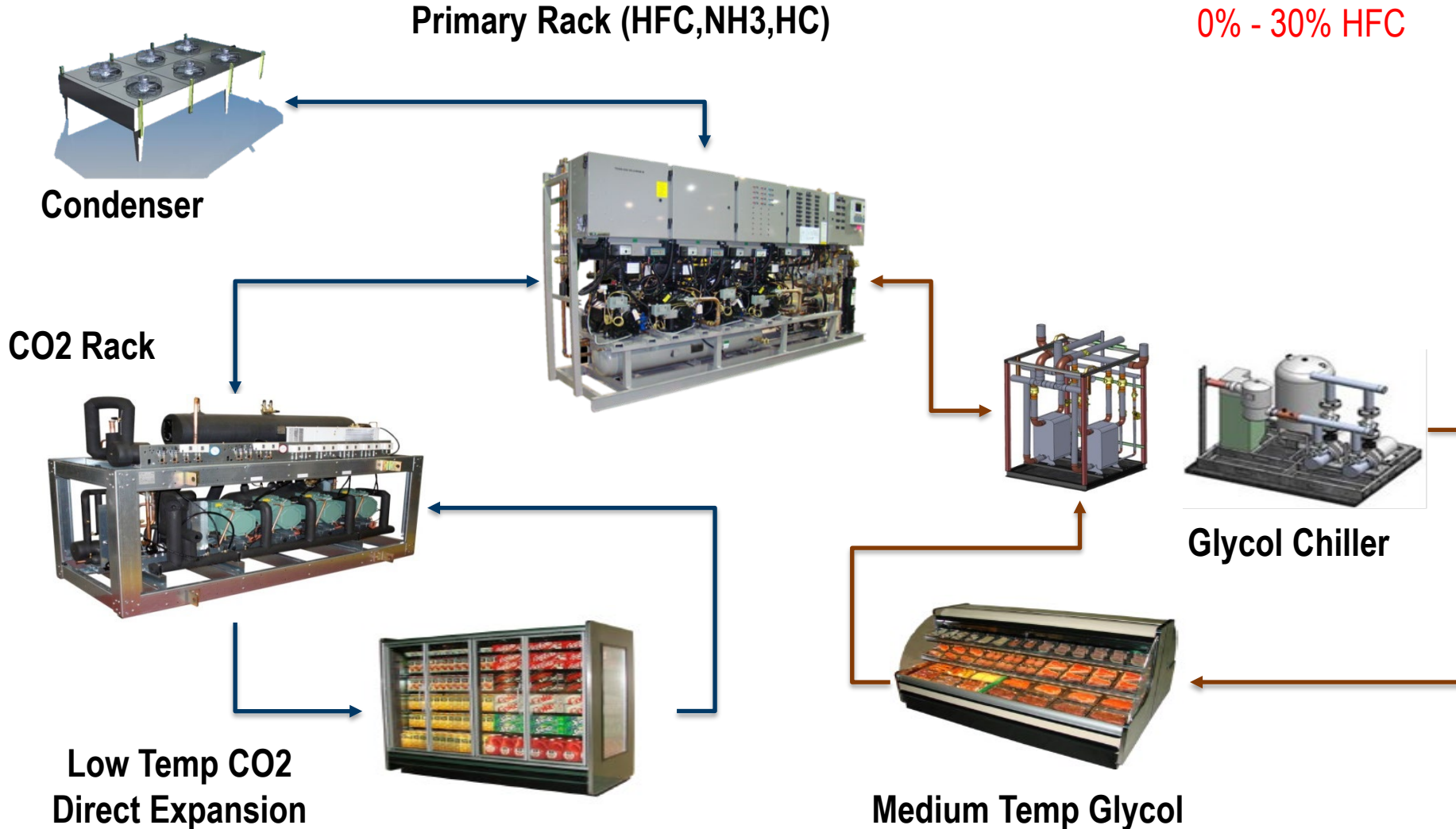
Low Temp CO2  
Direct Expansion



Medium Temp Glycol



Glycol Chiller

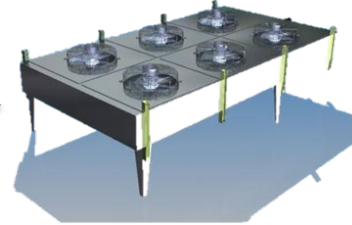


# SYSTEM SOLUTIONS

CASCADE SYSTEM CO2 MT LOF / CO2 LT DX



Primary Rack (HFC,NH3,HC)



Condenser

0% - 30% HFC



CO2 Hybrid Rack

Low Temp CO2  
Direct Expansion



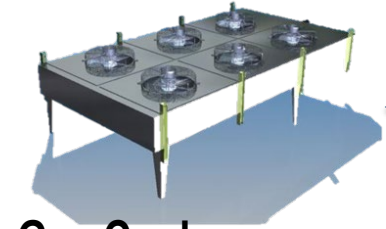
Medium Temp  
CO2 Liquid Overfeed  
(CO2 Pumped)



# SYSTEM SOLUTIONS

TRANSCRITICAL BOOSTER - CO2 MT DX / CO2 LT DX

## CO2 Transcritical Booster System 0% HFC



**Gas Cooler  
/ Condenser**



**CO2 Booster Rack**

**Low Temp CO2  
Direct Expansion**

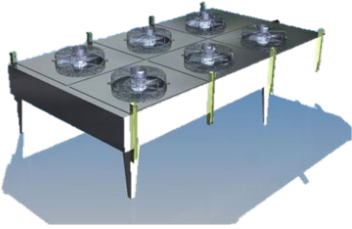


**Medium Temp CO2  
Direct Expansion**



# SYSTEM SOLUTIONS

HFC DIRECT EXPANSION MT DX, LT DX



**Condenser**



**HFC Rack**

100% HFC

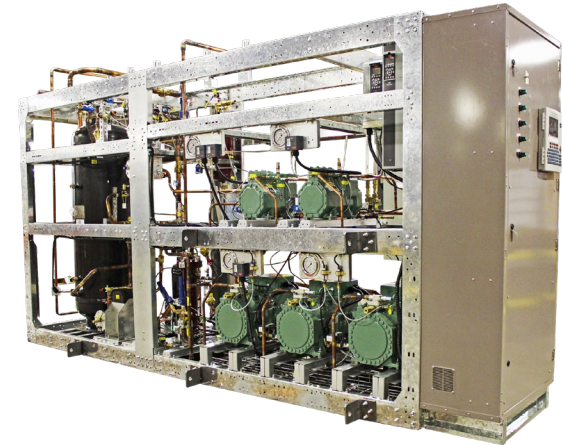


**Low Temp HFC  
Direct Expansion**

**Medium Temp HFC  
Direct Expansion**

### CO2 TRANSCRITICAL BOOSTER

- Comparable to HFC concept
- Single refrigeration system
- Single refrigerant (CO2)
- HFC free
- Higher operating pressures
  - Gas cooler 700 – 1500 psi
  - Flash gas tank 500 – 600 psi
  - MT suction group 380 – 450 psi
  - LT suction group 180 – 220 psi
- Applications
  - LT only – Not common
  - MT only – Common use
  - Booster (LT & MT) – Most common application

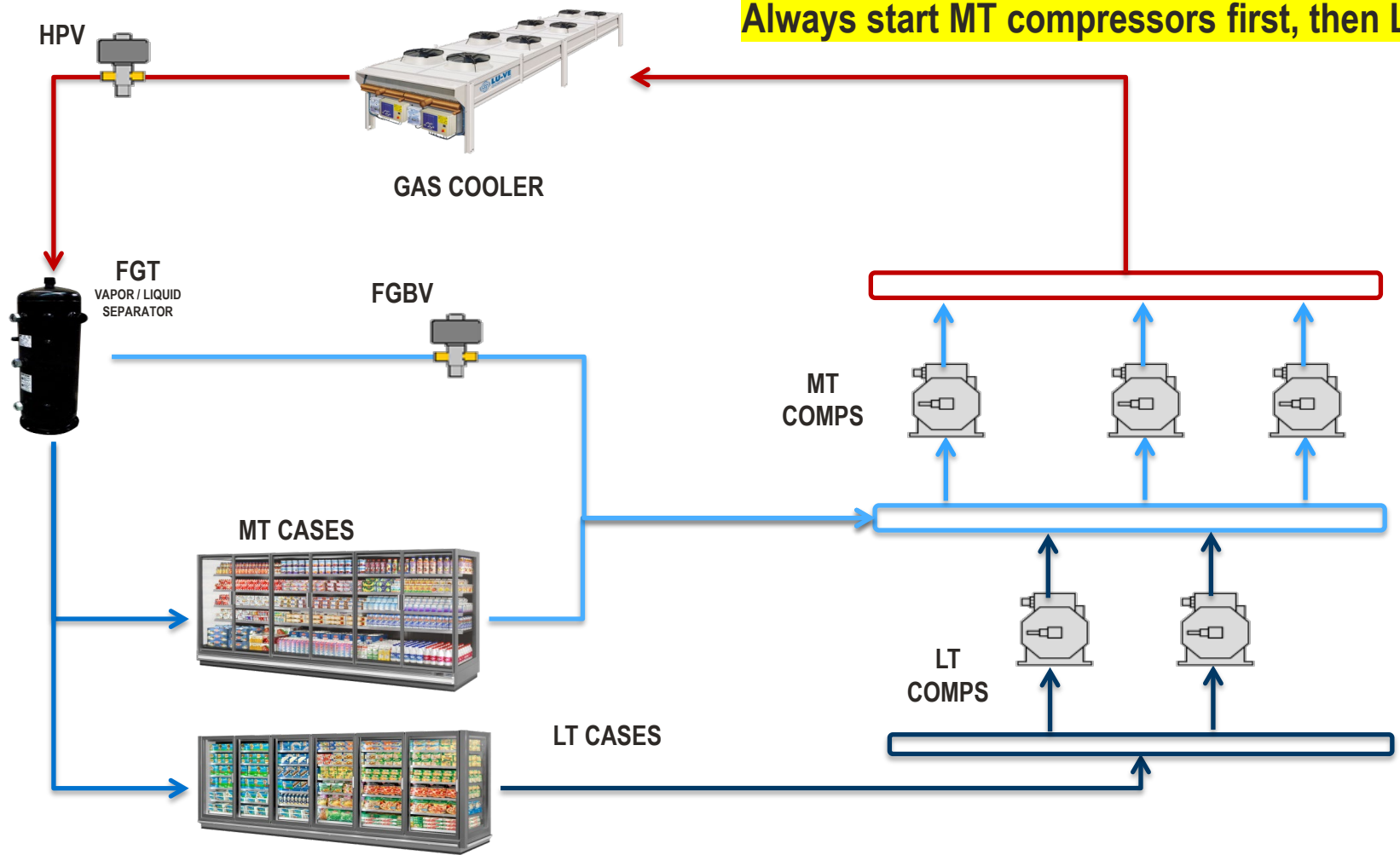


CO2 TRANSCRITICAL BOOSTER RACK

# CO2 TCB REFRIGERATION SYSTEM

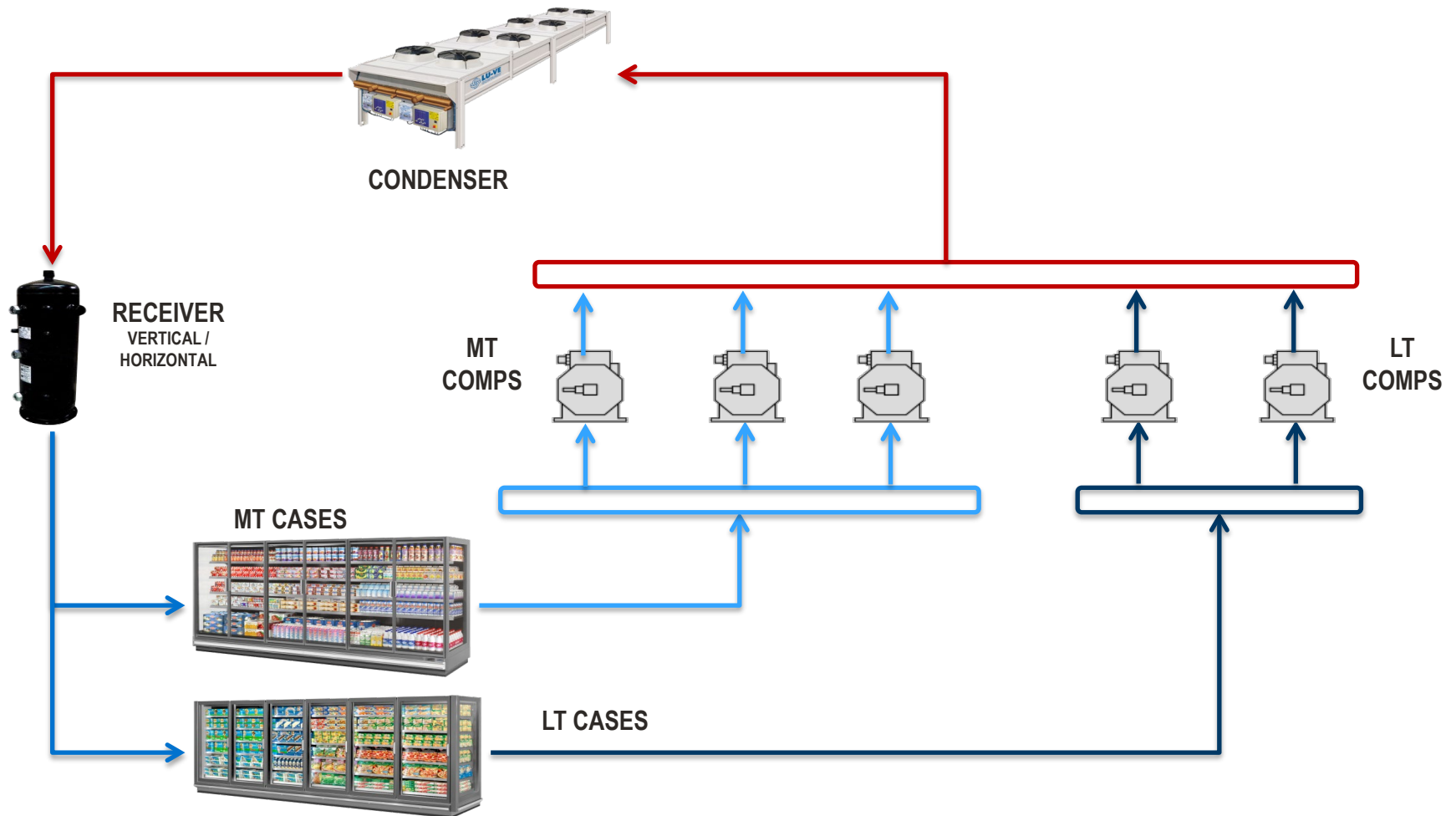
SIMPLE PIPING REPRESENTATION – MAIN COMPONENTS

Always start MT compressors first, then LT



# HFC REFRIGERATION SYSTEM

SIMPLE PIPING REPRESENTATION – MAIN COMPONENTS



# HFC VS CO2 TRANSCRITICAL SYSTEM

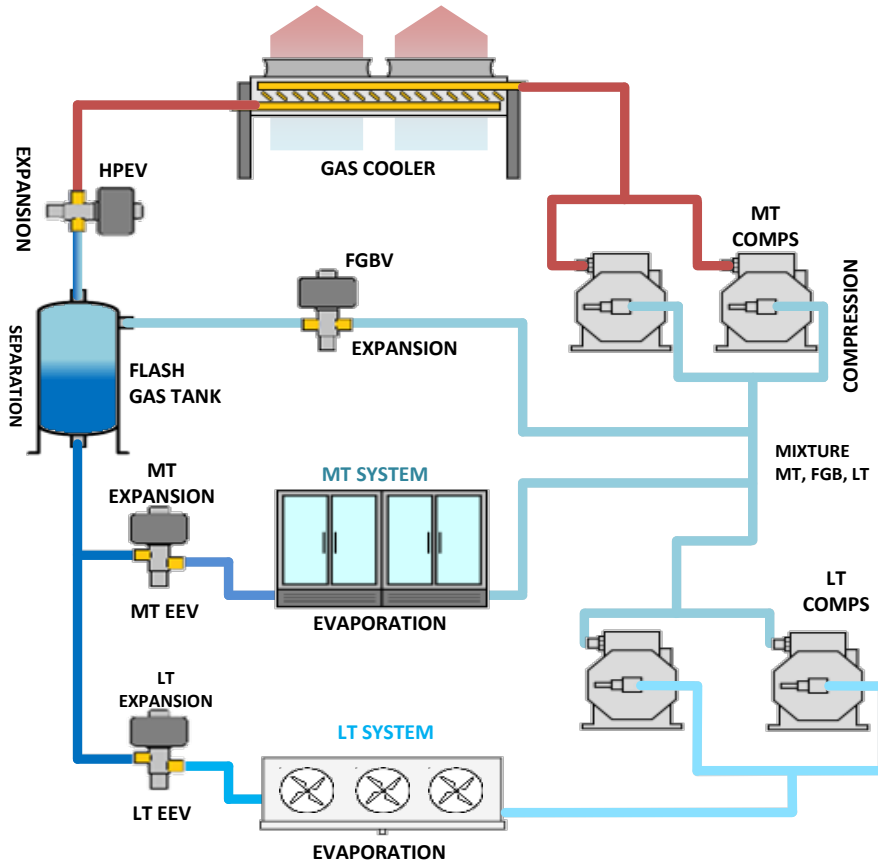
## SUMMARY

ITEM	HFC	Standard CO2 TCB
MT Compressor discharge	Oil separator / Condenser	Oil separator / Gas cooler (Condenser)
LT Compressors discharge	Oil separator / Condenser	Suction of MT Compressors ( <b>Important for startup</b> )
Receiver / Flash Tank	Storage liquid refrigerant	Separates liquid and vapor
Receiver / Flash Tank pressure	Similar as condenser pressure	Few hundred psi lower than gas cooler
System main pressure levels	3 (LT suction, MT Suction, Condenser)	4 (LT suction, MT Suction, Flash tank, Gas cooler)
High Pressure Valve	N/A	Manages gas cooler pressure Reduces pressure from GC to FGT,
Flash gas bypass valve	N/A	Manages flash tank pressure Removes vapor from FGT to MT compressor suction
Piping material	K and L type copper	K, L & High-pressure copper (K65 / XHP) or SS pipe
Operating pressures	5 - 450 psig	150 – 1450 psig (approximately)
Power outage	No problem <b>Passive system</b>	Could loose charge depending on ambient temperature <b>Active system</b>



# DESIGN OVERVIEW

## TC BOOSTER – FLASH GAS BYPASS



HPEV – High Pressure Expansion Valve  
FGBV – Flash Gas Bypass Valve

## FLASH GAS

- Is recirculated in the system (MT comps use lots of energy to recirculate)
- Does not produce cooling effect
- Flash gas formation increases with higher ambient temperatures or gas cooler outlet temperature
- Reduces pressure from FGT (520 psi) to MT suction pressure (420 psi) causing energy losses
- Can cause low superheat on MT compressors



## STANDARD CO2 TRANSCRITICAL BOOSTER SYSTEM

- Air cooled gas cooler – Basic design for cold weather locations

## HIGH AMBIENT CONDITION SOLUTIONS – HIGH PRESSURE SIDE

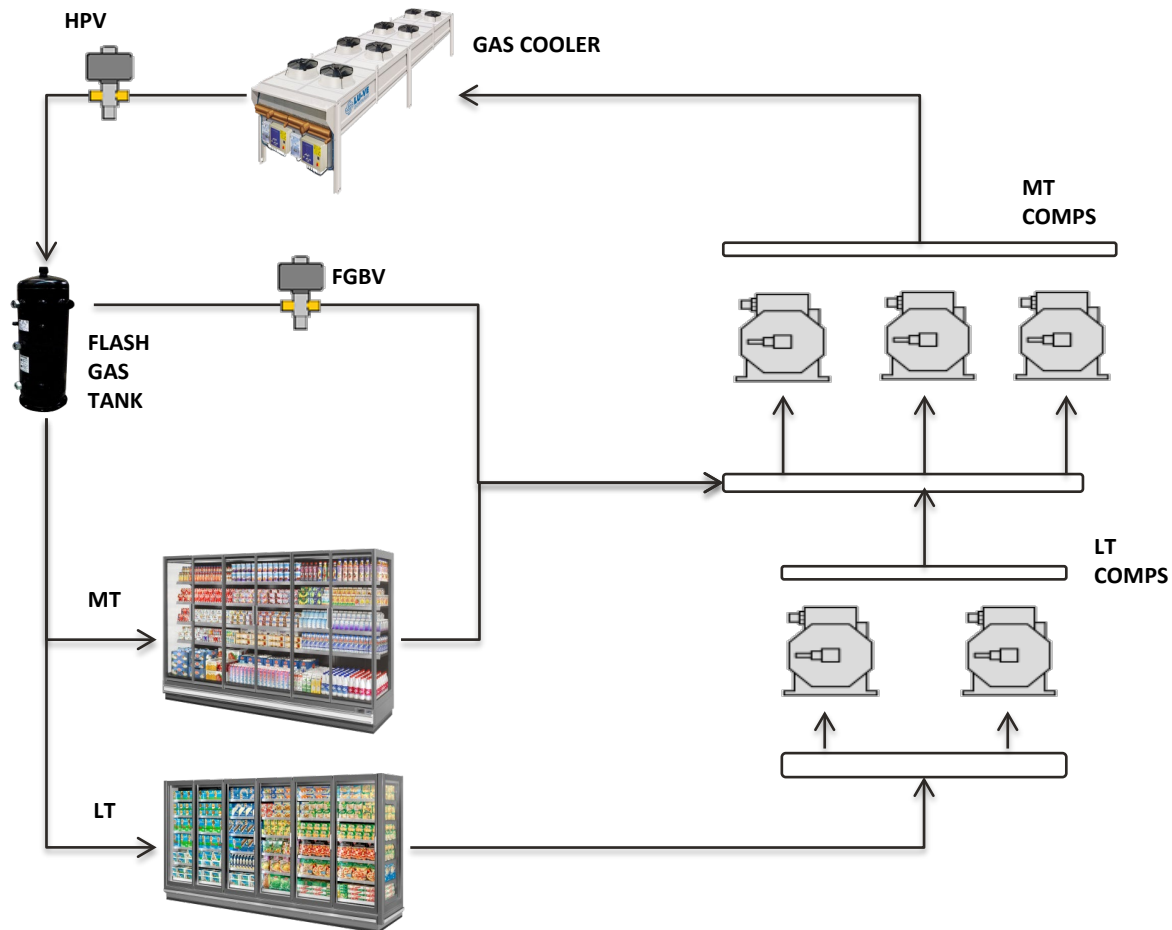
- Adiabatic cooling – Most commonly used (BAC and Guntner gas coolers)
- Mechanical Cooling – Not commonly used
- Parallel Compression – Few installations
- Ejectors + Parallel Compression – Not commonly used
  - High pressure ejectors
  - Low pressure ejectors
- ETE – Extreme Temperature Efficiency – Kysor Warren Epta

## ANY AMBIENT CONDITION SOLUTIONS – LOW PRESSURE SIDE

- FTE – Full Transcritical Efficiency – Kysor Warren Epta
- Liquid ejectors – Not commonly used

# DESIGN OVERVIEW

## STANDARD TRANSCRITICAL BOOSTER (TCB)



### Description

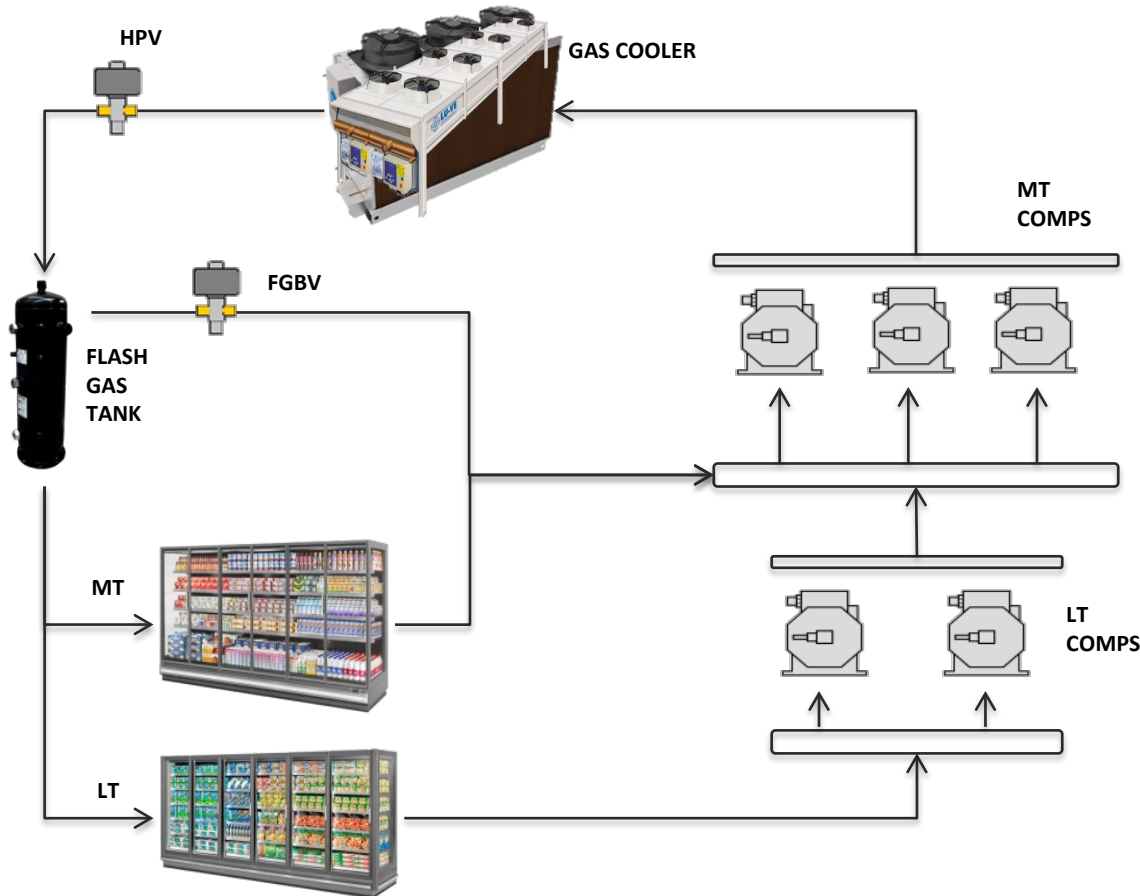
Basic design

Simple design

Loses efficiency on high ambient conditions

# DESIGN OVERVIEW

## ADIABATIC GAS COOLING



### Description

Water assisted gas cooler, increased efficiency during high ambient conditions.

Simple design, easy to implement

Improves efficiency in high ambient temperature conditions & low humidity

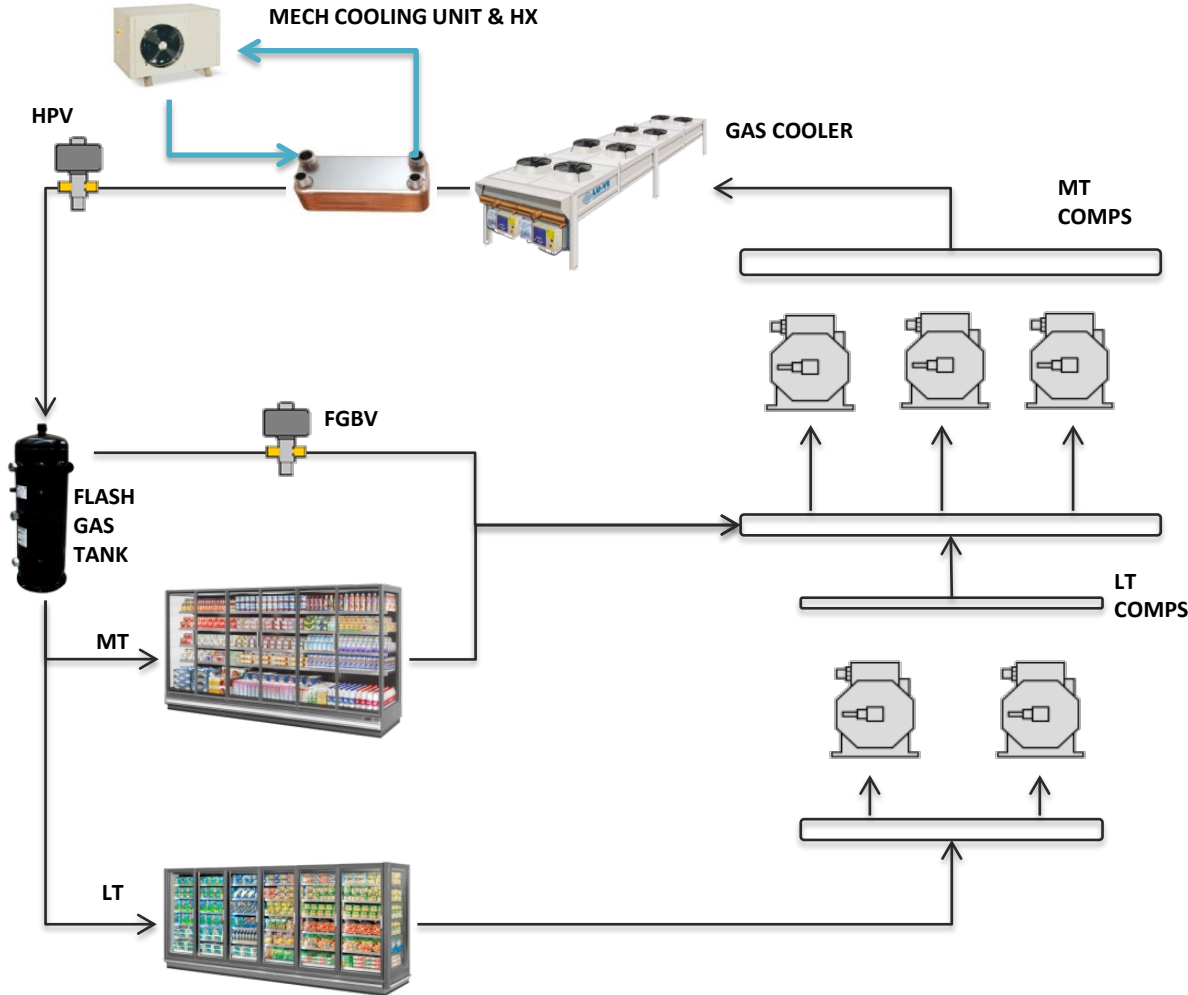
Higher up-front cost

Service factor needs to be considered

- Water usage
- Water cost
- Maintenance

# DESIGN OVERVIEW

## EXTERNAL MECHANICAL COOLING



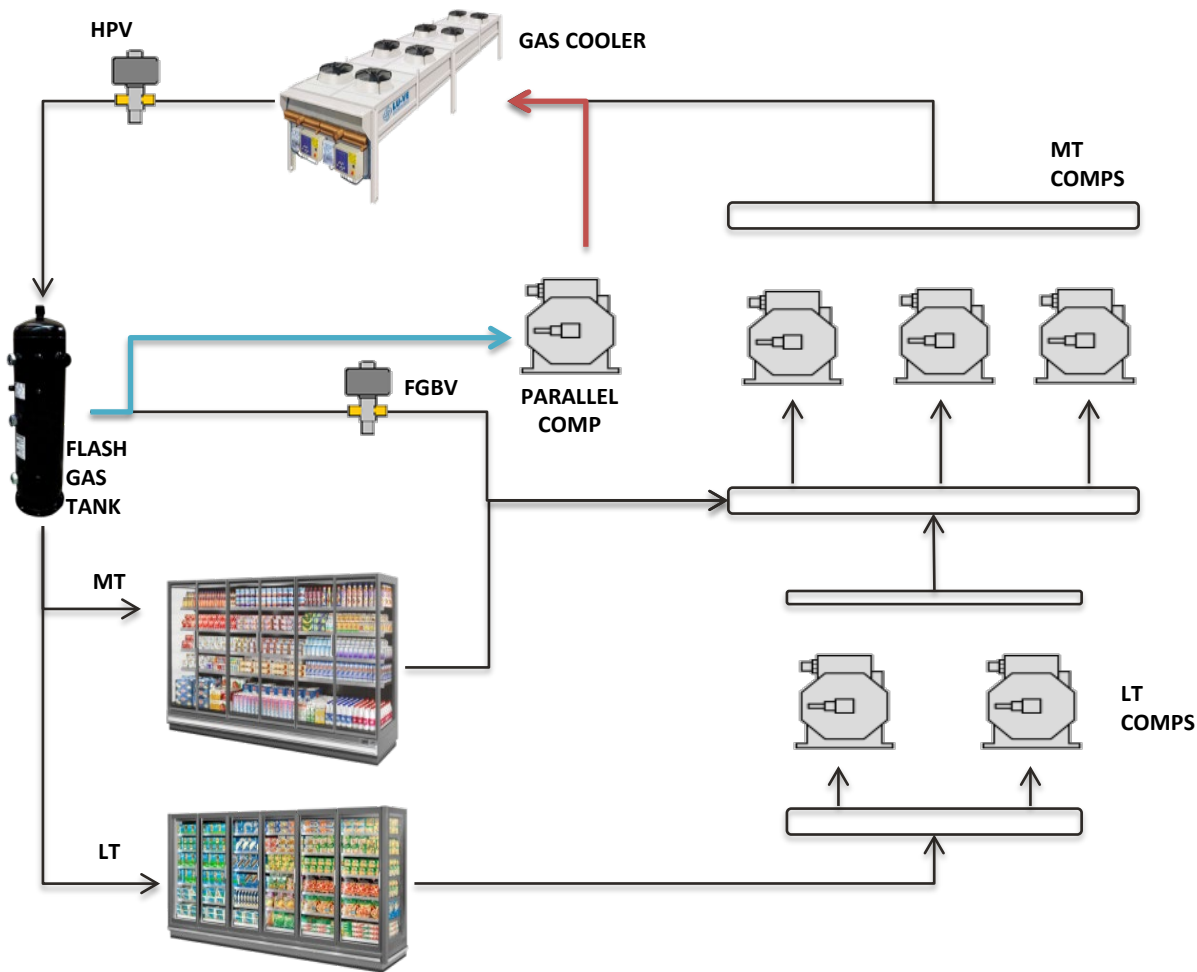
### Description

- Additional external condensing unit
- Cooling effect on CO<sub>2</sub> leaving the gas cooler
- Reduces flash gas formation
- Less work to remove flash gas from tank
- Improves efficiency

- Requires additional unit
- HFC / Natural refrigerants
- Maintenance

# DESIGN OVERVIEW

TCB + PARALLEL COMPRESSION



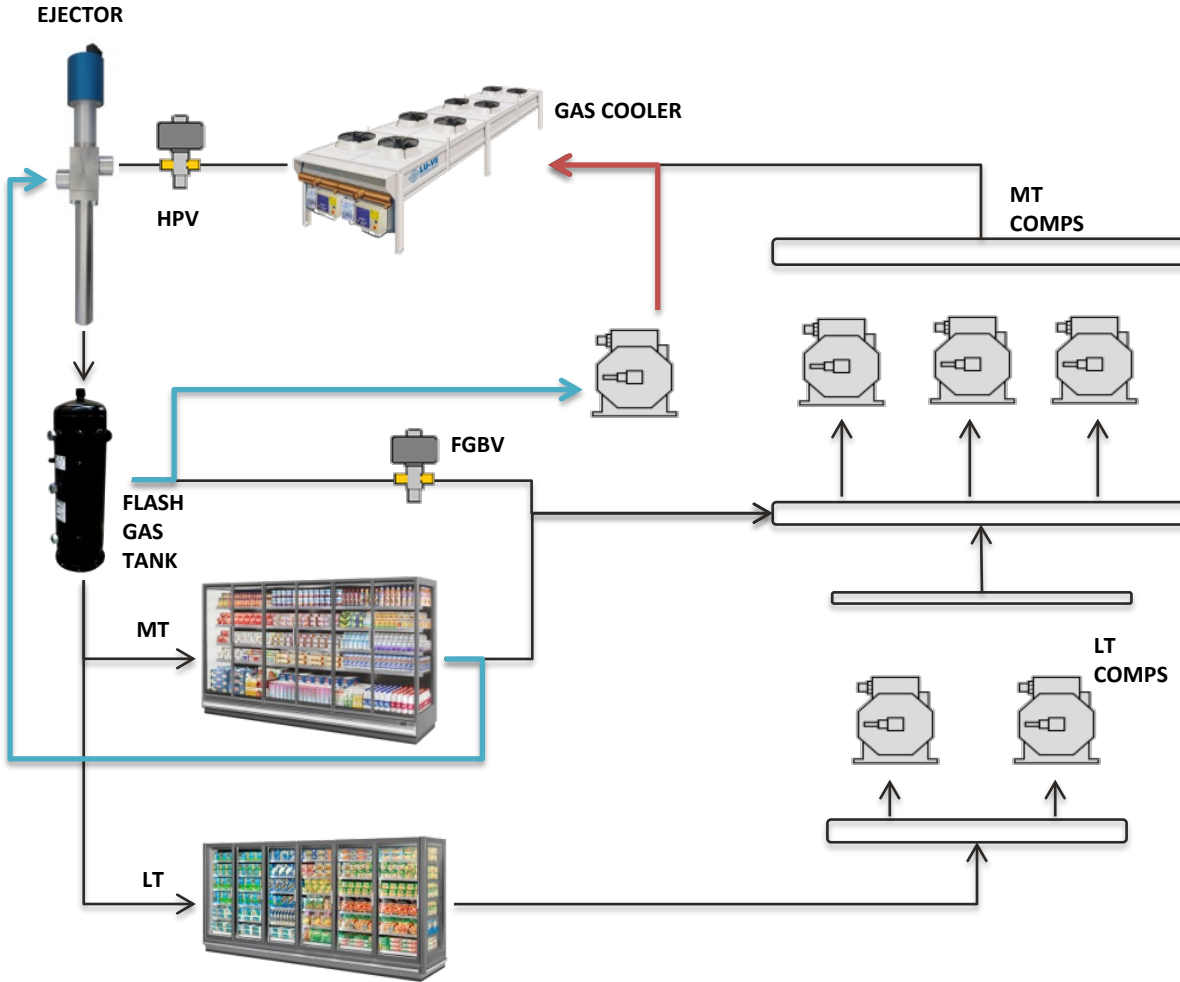
## Description

Removes the flash gas from the liquid receiver.  
Parallel compressor runs at flash tank pressure  
Requires less compression work than MT comps  
Higher efficiency than MT compressor

Additional control algorithms  
Additional VFD

# DESIGN OVERVIEW

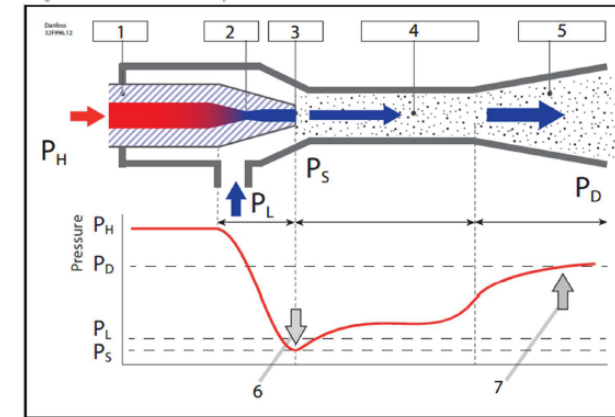
TCB + PARALLEL COMPRESSION + HIGH PRESSURE EJECTOR



## Description

Mechanical device to use the energy from the high pressure gas to lift the pressure of MT suction return and transfer flow from MT suction to flash gas tank.

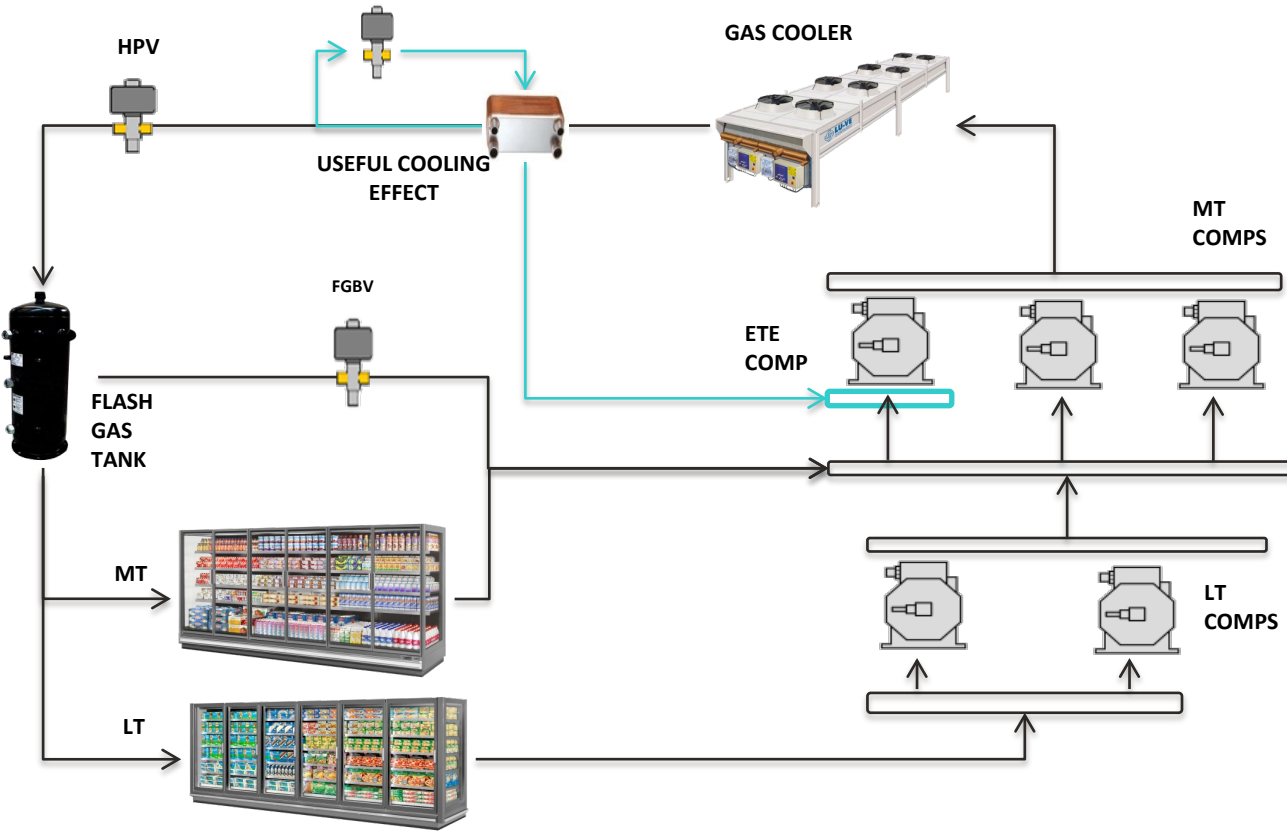
Requires the use of parallel compression on the same system  
Parallel compressor works more  
MT compressors work less



- |   |   |
|---|---|
| 1 | Nozzle  |
| 2 | Throat  |
| 3 | Exit  |
| 4 | Mixing chamber                                  |
| 5 | Diffuser  |
| 6 | Intake due to pressure differential             |
| 7 | Pressure increase due to reducing flow velocity |

# DESIGN OVERVIEW

EXTREME TEMPERATURE EFFICIENCY (ETE)



## Description

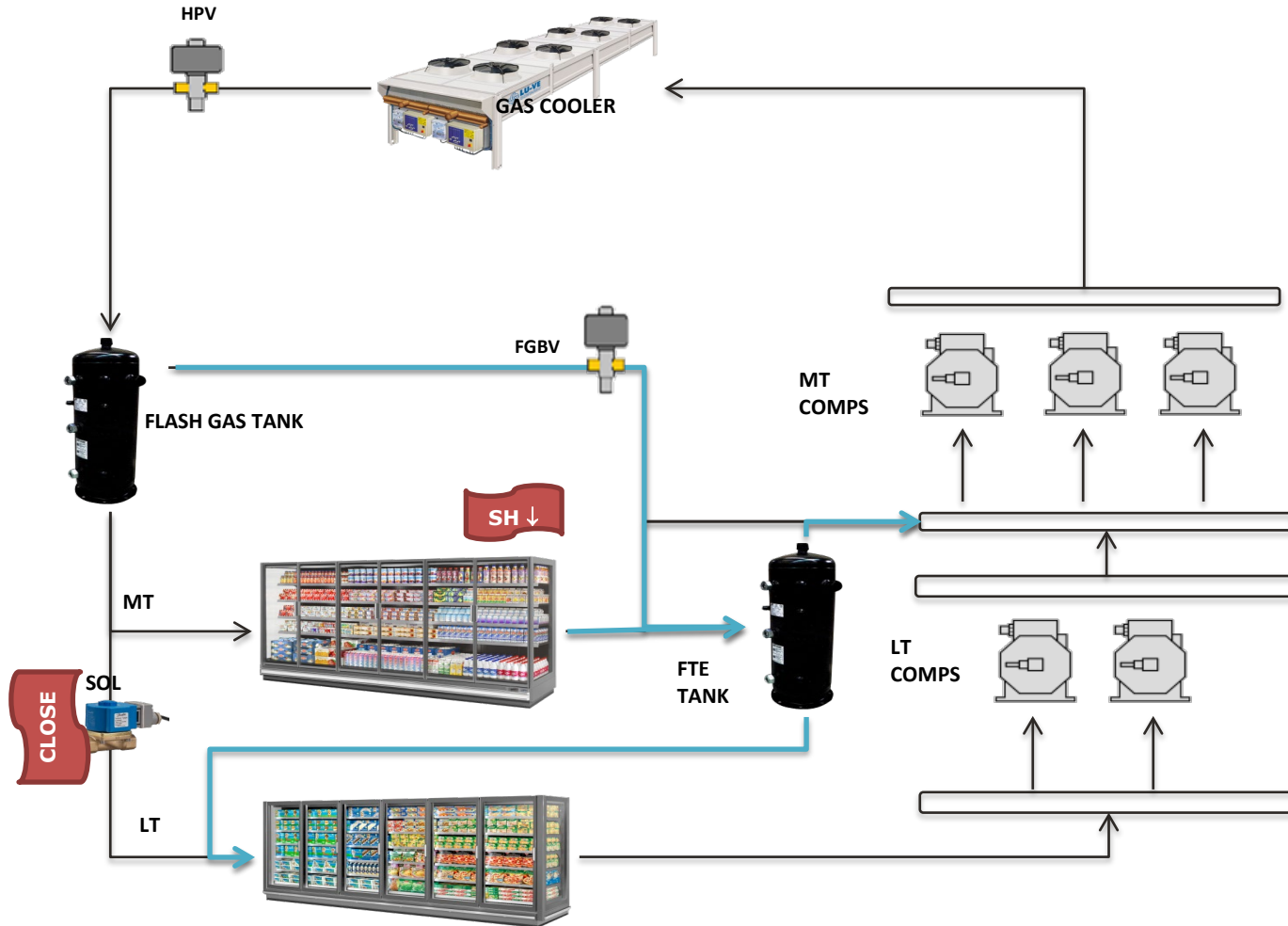
- Built in cooling system
- Cooling effect on CO2 after leaving the gas cooler
- ETE compressor runs at higher suction pressure compared to parallel compressor
- Reduces flash gas formation
- Less work to remove flash gas from tank
- Improves efficiency
- Uses CO2 from same system
- No need for adiabatic gas cooler

Additional suction group  
Additional VFD

**+200 Installations globally**

# DESIGN OVERVIEW

FULL TRANSCRITICAL EFFICIENCY (FTE)



## Description

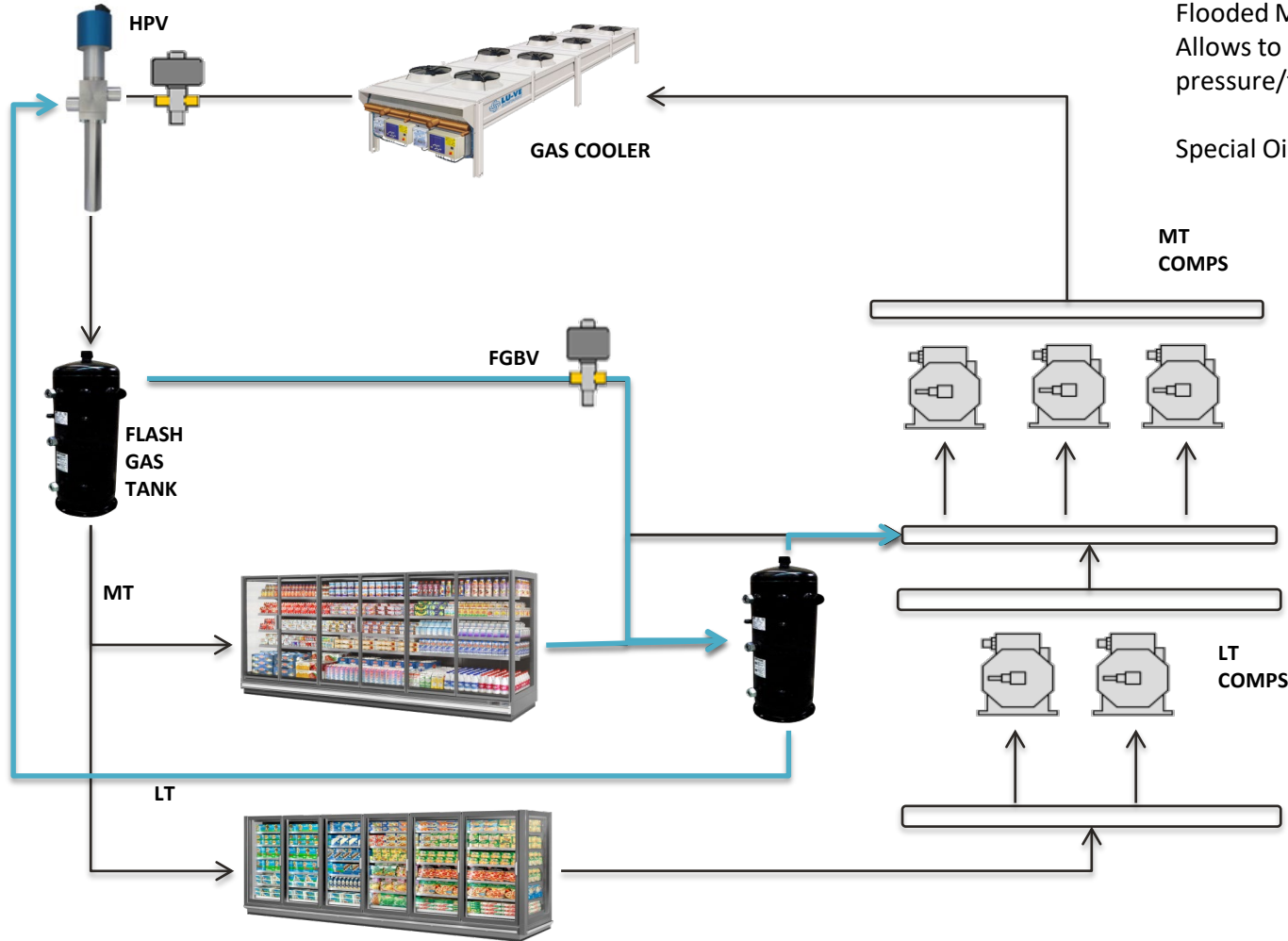
- System operates in the low pressure side
- Flooded MT cases with lower SH
- Increase MT cases capacity/efficiency
- Allows to increase MT compressors suction pressure/temperature
- Reduce energy consumption
- Independent of ambient temperature
- System requires MT and LT loads

**+600 Installations globally**



# DESIGN OVERVIEW

## LIQUID EJECTORS



### Description

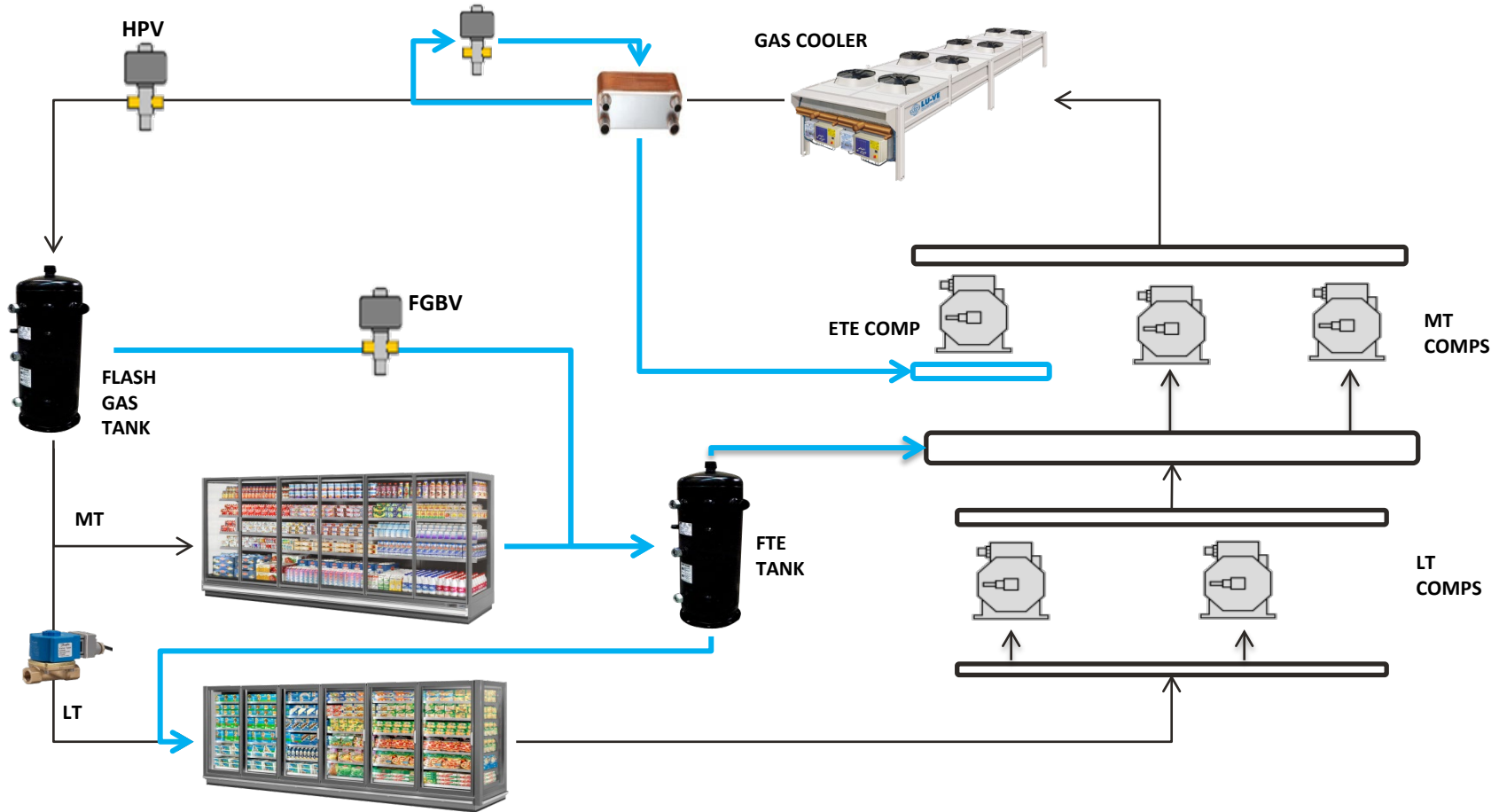
System operates in the low pressure side  
Flooded MT cases  
Allows to increase MT compressor suction  
pressure/temperature

Special Oil control logic

# CO2 TRANSCRITICAL SYSTEM

COMBINING TECHNOLOGIES FOR HIGHER EFFICIENCY

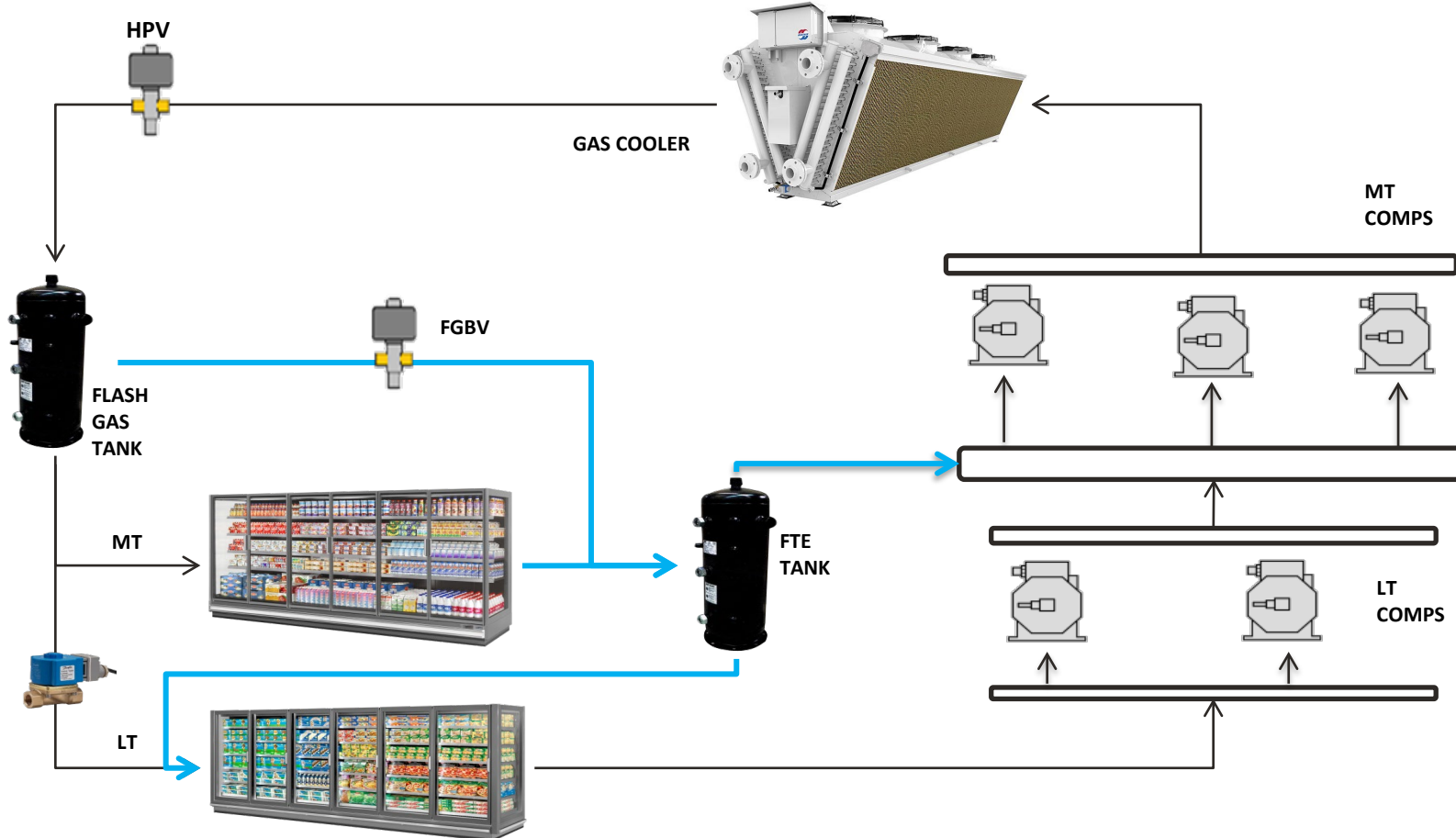
## EXTREME TEMPERATURE EFFICIENCY + FULL TRANSCRITICAL EFFICIENCY



# CO2 TRANSCRITICAL SYSTEM

COMBINING TECHNOLOGIES FOR HIGHER EFFICIENCY

## ADIABATIC GAS COOLING + FULL TRANSCRITICAL EFFICIENCY



# SUBSYSTEM & COMPONENTS



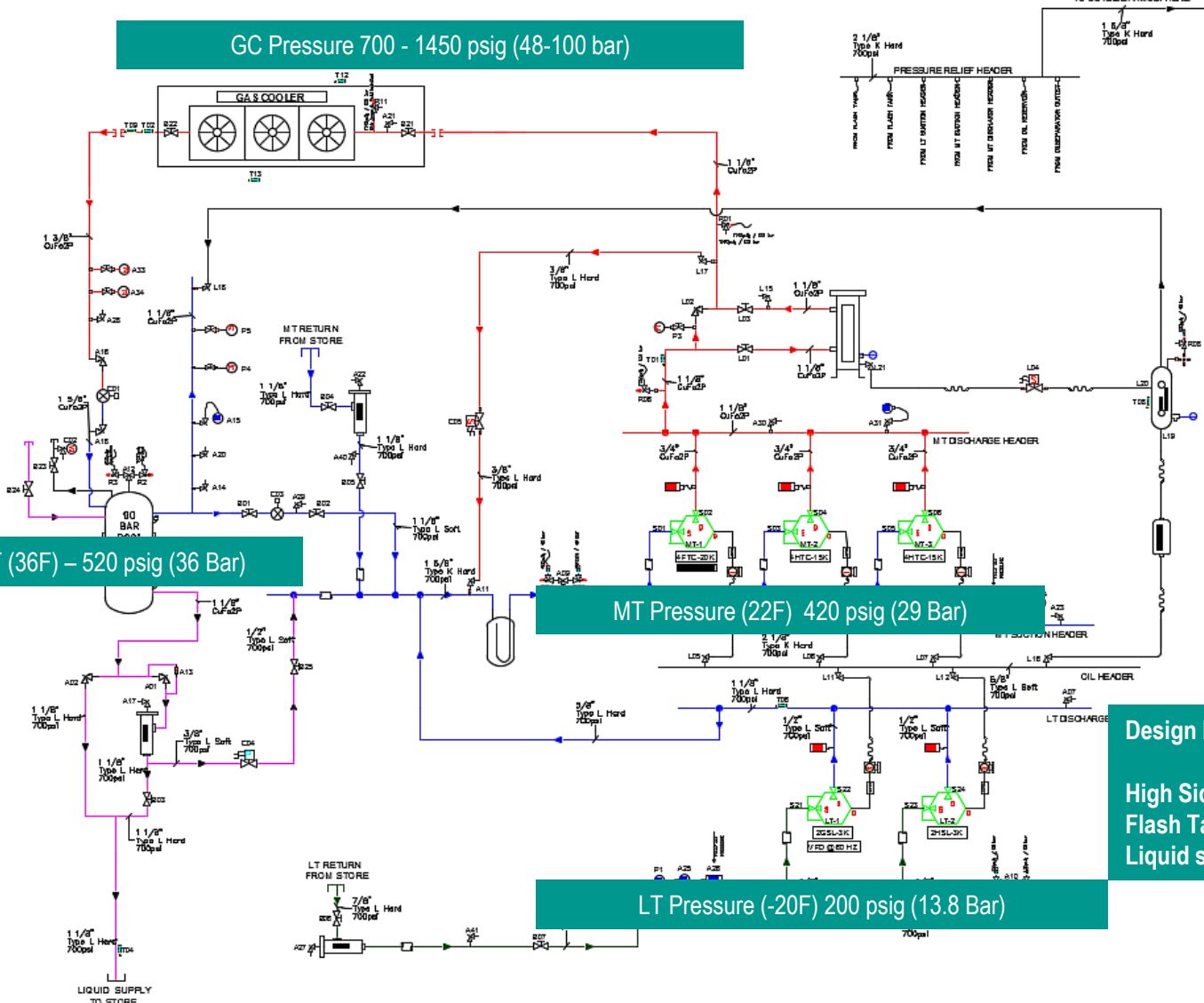
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GC Pressure 700 - 1450 psig (48-100 bar)

FGT (36F) - 520 psig (36 Bar)

MT Pressure (22F) 420 psig (29 Bar)

LT Pressure (-20F) 200 psig (13.8 Bar)



**Design Pressure**

- High Side 1740 psi
- Flash Tank - 870 or 1305 psi
- Liquid supply lines 655 psi

# RACK OVERVIEW

## SUBSYSTEM AND COMPONENTS

### RACK DESIGN OVERVIEW

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- System management
  - Gas cooler
    - Pressure
    - Fan operation
  - Flash gas tank pressure
  - Oil
    - Low pressure
    - High pressure
  - Superheat
  - Refrigerant

## High Pressure Valve

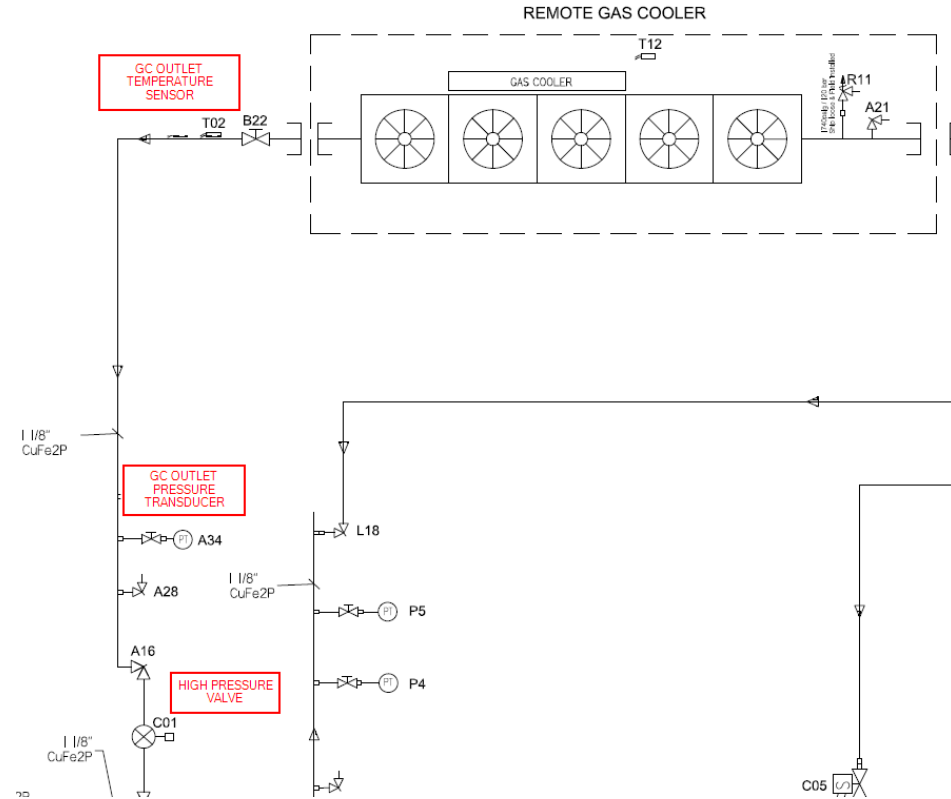
- Manages gas cooler pressure in transcritical operation mode for optimum energy efficiency
- Reduces CO2 pressure coming out of the gas cooler to the flash tank pressure levels
- Operating pressures 700-1450 psi

## Gas Cooler Outlet Temperature Sensor:

- Control of Gas Cooler Fan Speed
- Controller calculates optimum pressure
- Should be mounted as close as possible as gas cooler outlet header

## Gas Cooler Outlet Pressure Transducer

- Control opening of the High Pressure Valve to operate at highest efficiency
- Installed close to the High Pressure Valve



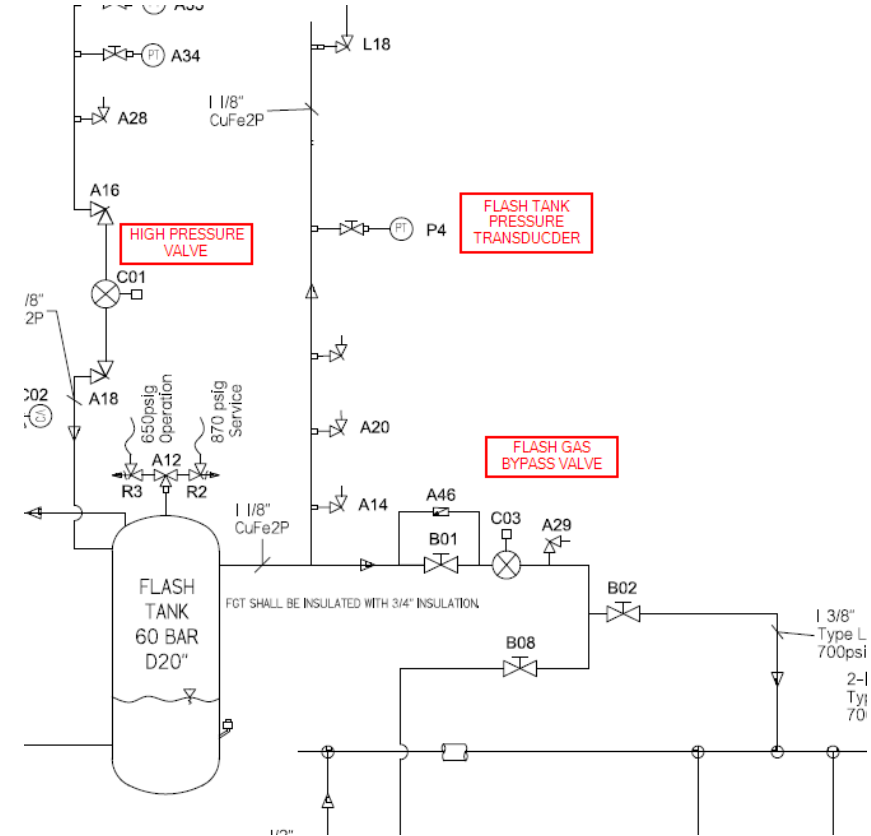


## Flash Gas Bypass Valve

- Maintains flash tank pressure to a specified set point.
- Allows vapor flow from flash tank to medium temperature compressor suction line.
- Operating pressure below 652 psig

## Flash Tank Pressure Transducer

- Control opening of the Flash Gas Bypass Valve.



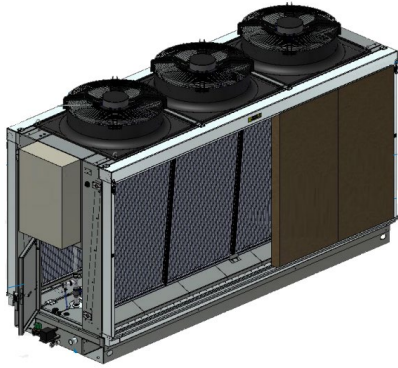
## MANUAL MANIPULATION

# SMA-12 SPORLAN STEP MOTOR ACTUATOR



<https://www.parker.com/content/dam/Parker-com/Literature/Sporlan/Sporlan-pdf-files/Sporlan-pdf-100/SD-213-SMA-12.pdf>

## Adiabatic (Hybrid) Gas Cooler



## Main Features

Design pressure: 1885 – 2030 psi

Fan motors: Variable speed (ECM)

Tube material: SS or High pressure copper

## Air Cooled Gas Cooler

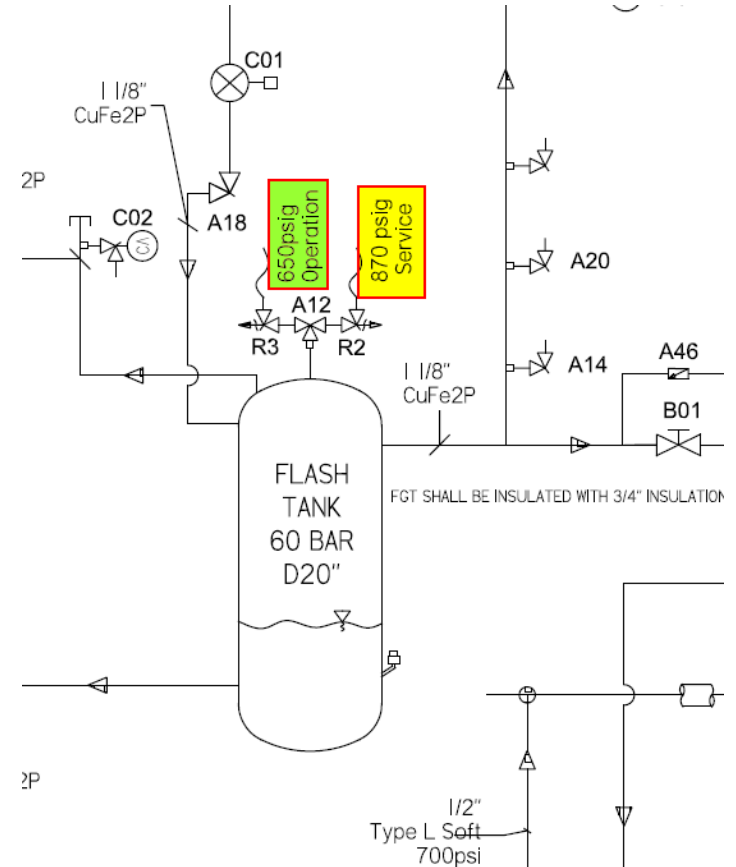
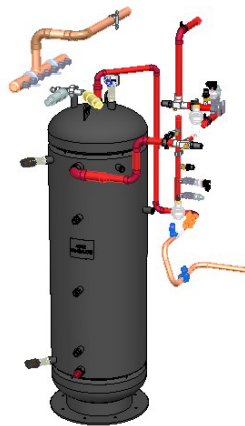


# FLASH TANK

- Separates liquid and vapor refrigerant phases so that only the liquid is sent to the evaporators and the vapor can be redirected to the MT compressor suction
- Holds the refrigerant charge for proper system operation
- Design pressure: 655, 870 or 1305 psi
- Dual pressure relief valves
  - Service valve only active during service or pump down
  - Service valve is manually activated through 3 way change over valve

Recommended level to be between first and second sight glasses during operation.

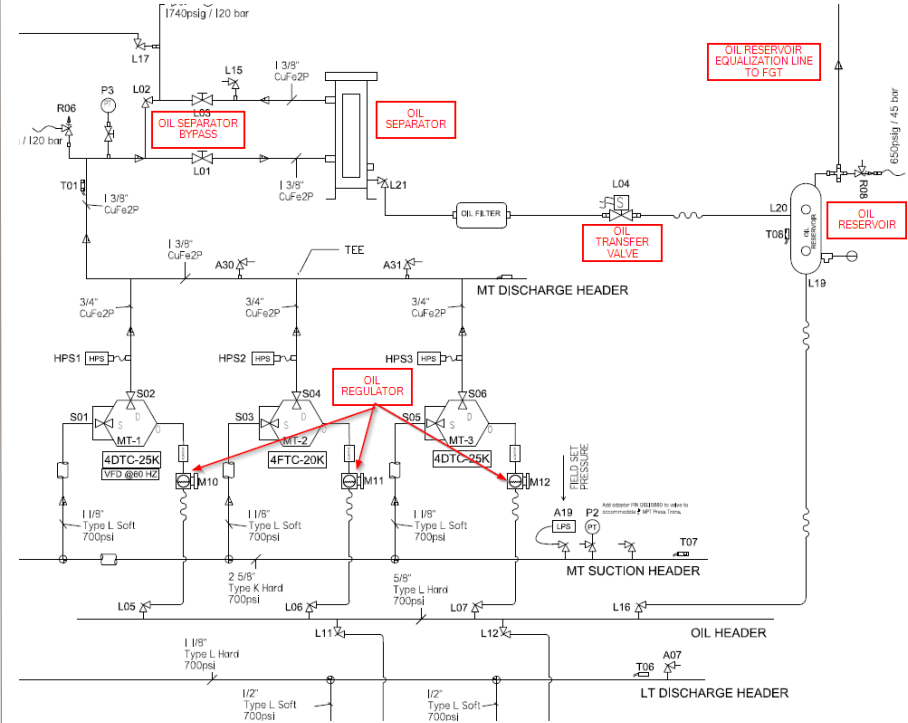
During pump-down the level should never exceed the tank's top sight glass.



- Ensures positive pressure in oil system to inject oil in LT & MT compressors
- Main components
  - Oil separator
  - Transfer valve (solenoid valve)
  - Oil reservoir
  - Oil regulator
  - Equalization line
  - Oil level sensor – if applicable
  - Oil separator bypass – if applicable

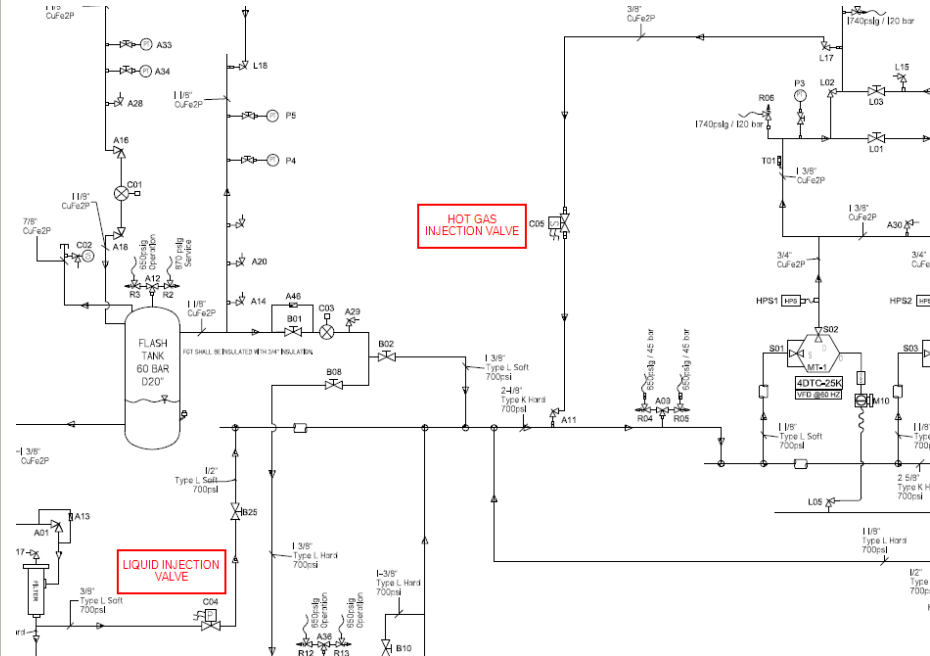


## LOW PRESSURE OIL SYSTEM



# SUPERHEAT CONTROL SYSTEM

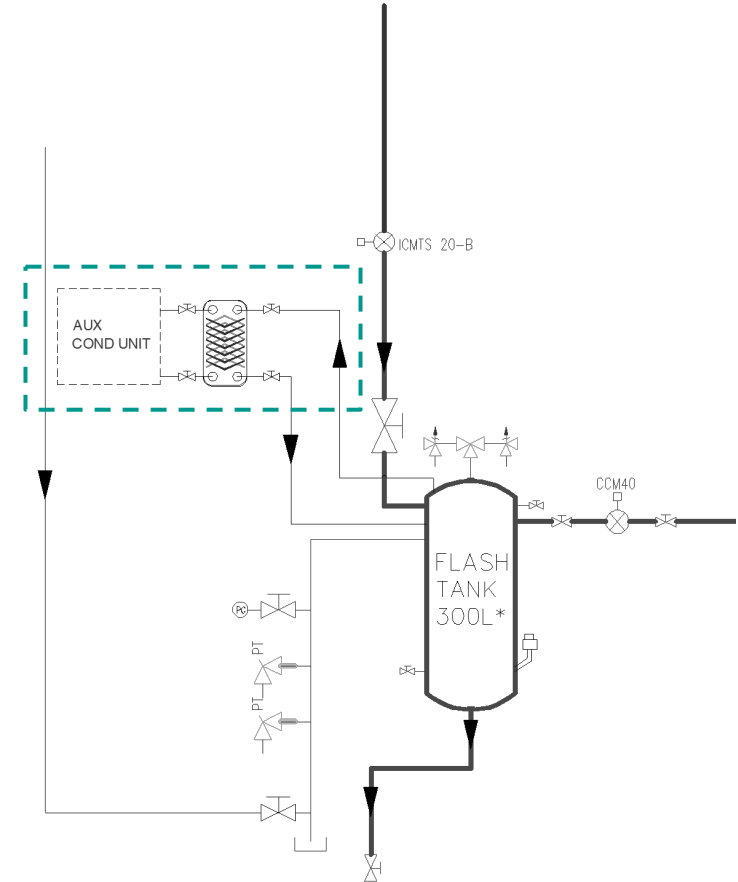
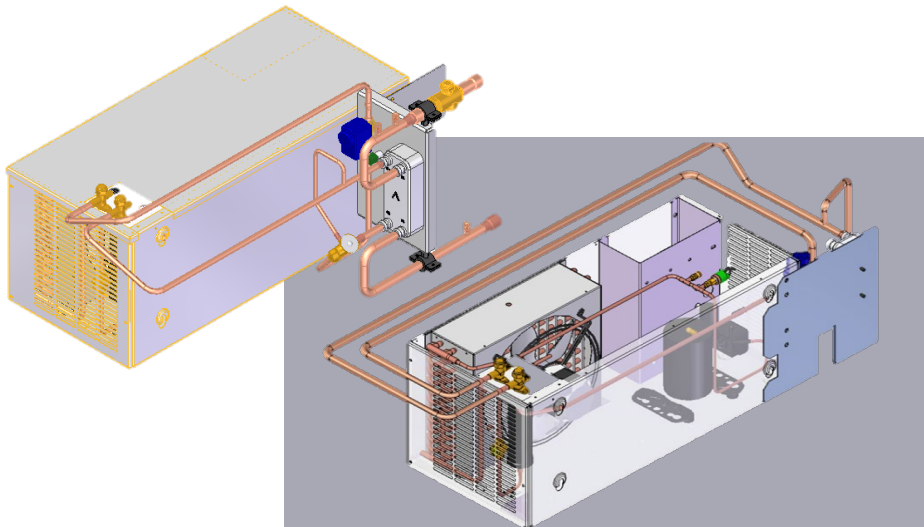
- Ensures proper superheat value for MT compressors
- Main components
  - Option 1
    - Hot gas injection valve (solenoid)
    - Liquid injection valve (EEV)
  - Option 2
    - Heat exchangers



# AUXILIARY CONDENSING UNIT (OPTIONAL)

## SUBSYSTEM AND COMPONENTS

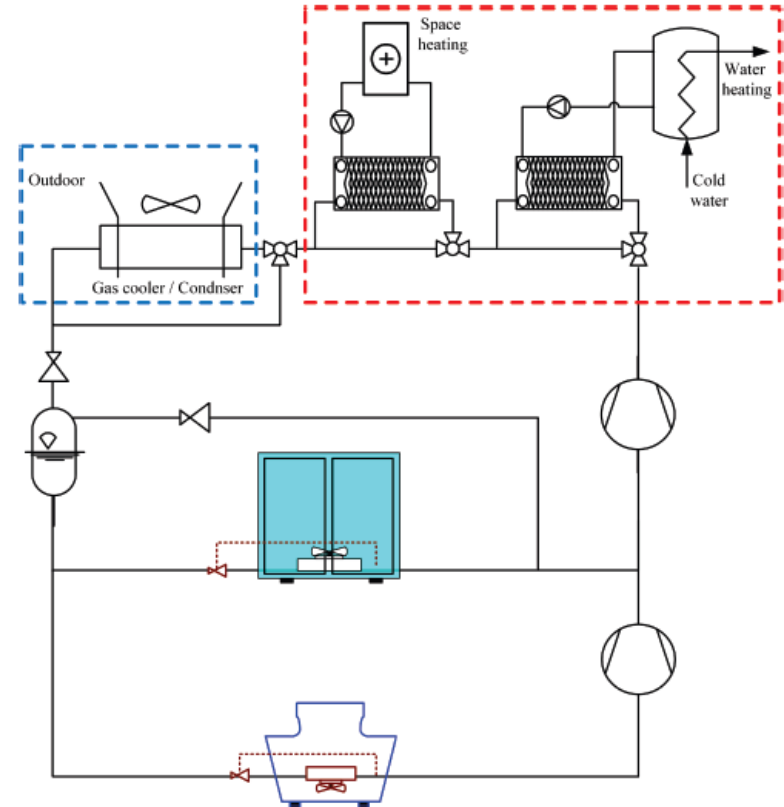
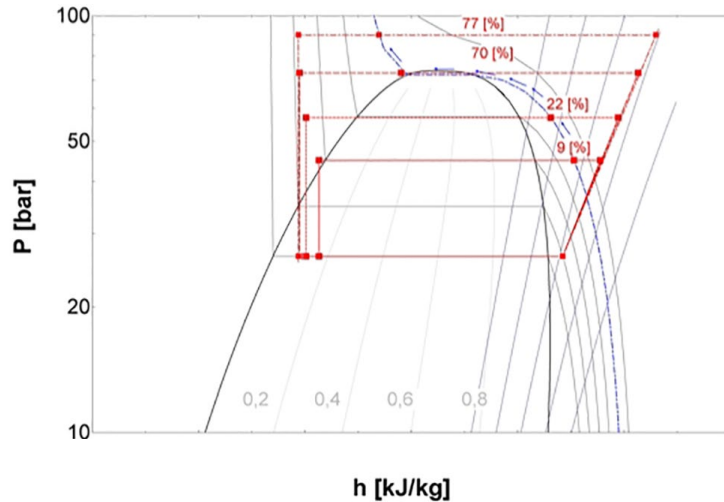
- Keep the pressure at the flash tank under relief valve setting to avoid loss of charge in power failures or servicing situations
- Condense CO<sub>2</sub> gas from flash tank
- Needs alternate power source to function under power failure situations



# HEAT RECLAIM (OPTIONAL)

## REFRIGERANT MANAGEMENT MODULE

- Recover energy heat that is normally rejected or wasted to the ambient
- Reduce the energy consumption needed for hot water, store heating or dehumidification purposes
- Heat reclaim capacity varies with system operating conditions, subcritical or transcritical modes.
- Direct or Indirect methods





# CONTROLLERS



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- Site manager
- Compressors control system
- HPV & FGBV control
- Case controller
  - Display cases
  - Evaporator coils
- Electronic Expansion Valves

# CO2 TRANSCRITICAL SYSTEMS

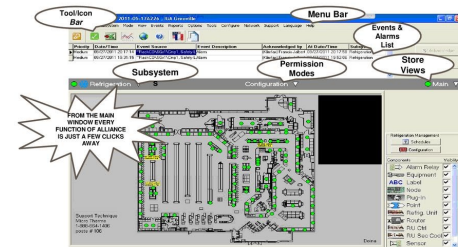
## SITE MANAGERS

- Manage communication between rack system and case controllers, defrost schedule, lights, HVAC in some cases

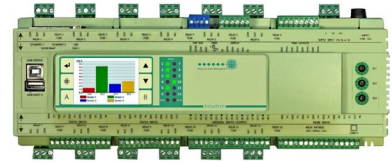


### Software – MT Alliance

Familiar Windows user interface:



- Manage compressor operation and all features in the rack
  - One or Two suction groups (LT & MT)
  - Variable capacity for lead compressors
  - Parallel compression (check with your controls provider)
  - Ejectors (check with your controls provider)



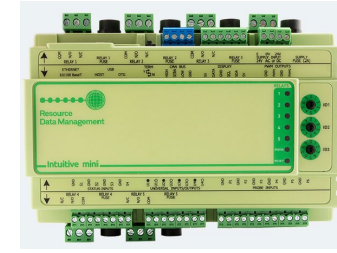
- Manage operation of high pressure valve and flash gas bypass valve in both transcritical and subcritical conditions
- Every controls manufacturer has its own control algorithm
- Some rack controllers have HPV and FGBV controls built in
- Check proper wiring for HPV and FGBV, per controller manufacturer



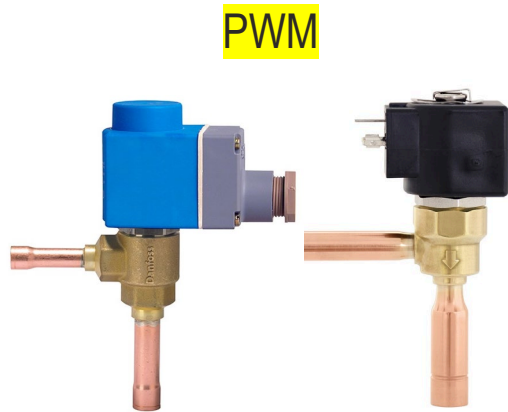
# CO2 TRANSCRITICAL SYSTEMS

## CASE CONTROLLERS

- Case controller are used in display cases as well cold rooms (coolers & freezers)
- Evaporator fan control
- Defrost control
- Lighting control
- Dual temperature
- Anti-sweat control
- EEV control
  - EEV's are required for CO2 control
  - Stepper or pulse valves available



- Meters the refrigerant flow going to the evaporator
- Modulates
  - To maintain air temperature set point (discharge for cases / return for walk-ins)
  - Superheat



\* In case of power failure:

Stepper valve will remain in its current position. Additional solenoid valve may be needed to close liquid refrigerant supply to avoid flooding compressors.

Pulse valve will fail closed – no additional solenoid valve is required

- Components used rated for proper operating pressures depending the use in the system
- Install PRV where needed (liquid or vapor rated valves)
- Reverse acting operation on gas cooler fans
  - 10V = OFF , 0V = ON (100%)
- Backup power for controls
- Refrigeration load management protocols
- High pressure switches at compressor discharge



# THANK YOU!!

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